

**STUDY OF NATURAL CONVECTION HEAT TRANSFER IN
A RECTANGULAR ENCLOSURE FROM ONE COOLED
SIDE WALL**

by
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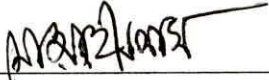
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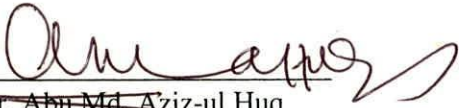
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MD. MOSTOFA HOSSAIN

Dedicated
To
My Beloved and Respected
Father Amiruddin Pramanik

CONTENTS

	Page No.	
TITLE	i	
CERTIFICATE OF APPROVAL	ii	
DECLARATION OF THE CANDIDATE	iii	
DEDICATION	iv	
CONTENTS	vi	
ACKNOWLEDGEMENT	viii	
ABSTRACT	ix	
NOMENCLATURE	x	
LIST OF FIGURES	xi	
LIST OF TABLES	xiii	
CHAPTER-ONE		
INTRODUCTION		
1.1	Introduction of natural convection and its physical consideration	1
1.2	Natural convection in an enclosure	4
1.3	1.3 Effect of thermal stratification	5
1.4	Motivation behind the selection of present work	6
1.5	Present study	7
CHAPTER-TWO		
LITERATURE REVIEW		
2.1	Introduction	8
2.2	Overview of early work	11
2.3	Empirical correlations: Enclosures	13
	2.3.1 Rectangular cavities	13
CHAPTER-THREE		
MATHEMATICAL MODELING OF THE PROBLEM		
3.1	Description of the problem	17

3.2	Governing equations	17
3.3	Mathematical equations	21

CHAPTER-FOUR

EXPERIMENTAL SETUP

4.1	General description of the experimental facility	22
4.2	The test section	23
4.2.1	Rectangular enclosure	23
4.2.2	Thermoelectric module	23
4.2.3	Data acquisition system	24
4.2.4	Rectifier	25
4.2.5	Thermocouples	25
	a) Position of the thermocouples in the enclosure	25
4.3	Details of the cold plate assembly	25

CHAPTER-FIVE

EXPERIMENTAL PROCEDURE

5.1	Instrumentation	27
5.2	Calibration of thermocouple	27
5.3	Temperature measurement	27
5.4	Current and voltage measurement	28
5.5	Experimental procedure	28

CHAPTER-SIX

RESULTS AND DISCUSSIONS

6.1	Discussion on the results and findings	30
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CHAPTER-SEVEN

CONCLUSIONS AND RECOMMENDATION

7.1	Conclusions	35
7.2	Recommendations for future work	36

REFERENCES

37

APPENDIX-A: TABLES

APPENDIX-B: DATA

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ABSTRACT

An experimental investigation of natural convection heat transfer in rectangular enclosure from one cooled vertical side wall is carried out. In the experiment the side wall is cooled by a thermoelectric module (TEM). The measurements cover the temperatures of the cold plate and air in the enclosures at different interval of time at 14 different locations which are measured and recorded by data acquisition system. The power consumed by a thermoelectric module (TEM) during the transient and steady state conditions is also recorded. In an experiment lasting 1 hour, the enclosure air is cooled down to a minimum of 5°C, when the cold wall temperature is -4°C. With time, temperatures of both cold wall and air within the enclosure decrease asymptotically to a steady state value.

From the temperature distribution data, stratification within the enclosure is observed. The stratification indicator γ is estimated at three different horizontal distances from the cold plate for the time interval of 15, 25, 50, 60 and 80 minutes and found to be decreased with the time.

Heat transfer rate, heat transfer coefficient, Nusselt number and Rayleigh number are calculated from the measured experimental data during transient and steady state regime. The Rayleigh number and Nusselt number cover the range, 1.47×10^6 to 3.08×10^7 and 18.55 to 33.50 respectively.

From the experimental results a correlation is developed in terms of Nusselt number and Rayleigh number which is found to be capable of correlating all experimental data within $\pm 15\%$.

NOMENCLATURE

<u>Symbols</u>	<u>Meaning</u>	<u>Units</u>
g	gravitational acceleration	m/s^2
h	heat transfer coefficient	$w/m^2 K$
H	enclosure height	m
k	thermal conductivity	$w/m K$
L	enclosure length	m
N_u	Nusselt number	dimensionless number
P	Pressure	N/m^2
p_o	minimum density	
P_r	Prandtl number	dimensionless number
G_r	Grashof number	dimensionless number
R_{ah}	Rayleigh number	dimensionless number
T	Temperature	K
T_c	temperature of the cold plate	K
T_o	initial temperature of air	K
T_α	average temperature of the last column of the air	K
$T_{\alpha a}$	average temperature of twelve thermocouples	K
T_h	adiabatic temperature wall temperature of enclosures	K
T_1	Temperature of heated surface	K
T_2	Temperature of cold surface	
τ	Tilt angle	
ΔT	temperature difference, $T_h - T_c$	K
u	velocity vector, $\{u, v\}$	
U	dimensionless velocity vector $\{U, V\}$	
U, V	dimensionless velocities in X and Y coordinates	
x, y	Coordinates	
θ	dimensionless temperature	
μ	dynamic viscosity	$Kg/s m$
ν	kinematic viscosity	m^2/s
γ	Temperature gradient	$^\circ C/m$
ρ	Density	Kg/m^3
ρ_o	maximum density	
ψ	stream function	m^2/s
Ψ	dimensionless stream function	
α	thermal diffusivity	m^2/s
β	Coefficient of thermal expansion	K^{-1}
C_p	Specific heat at constant pressure	$J/kg ^\circ K$
m	Mass	Kg
Q	Heat transfer	W
q''	Heat flux	W/m^2
A	Area of cold plate	m^2
t	Time	S

LIST OF FIGURES

- Fig. 1.1 Conditions in a fluid between large horizontal plates at different temperatures (a) Unstable temperature gradient (b) Stable temperature gradient.
- Fig. 1.2a Boundary layer development on a heated vertical plate.
- Fig. 1.2b Boundary layer development on a cooled vertical plate.
- Fig. 1.3 Natural convection along a cold vertical wall environment.
- Fig. 2.1 Free convection in a rectangular cavity.
- Fig. 3.1 Schematic configuration of the problem and coordinate system.
- Fig. 4.1 Schematic diagram of the experimental facility
- Fig. 4.2 Position of the thermocouples in the enclosure
- Fig. 4.3 Schematic diagram of thermoelectric module with power supply
- Fig. 4.4 Details of cold plate assembly
- Fig. 5.1 Calibration of thermocouple
- Fig. 6.0 The effect of stratification in a cold vertical wall enclosure
- Fig. 6.1 Temperature at different height 'y' in the enclosure at different distance 'x' from cold plate after 15 mins
- Fig. 6.2 Temperature at different height 'y' in the enclosure at different distance 'x' from cold plate after 25 mins
- Fig. 6.3 Temperature at different height 'y' in the enclosure at different distance 'x' from cold plate after 50 mins
- Fig. 6.4 Temperature at different height 'y' in the enclosure at different distance 'x' from cold plate after 60 mins
- Fig. 6.5 Temperature at different height 'y' in the enclosure at different distance 'x' from cold plate after 80 mins
- Fig. 6.6 Effect of the enclosure stratification on heat transfer from air to the cold vertical wall (after 15 mins)
- Fig. 6.7 Effect of the enclosure stratification on heat transfer from air to the cold vertical wall (after 25 mins)
- Fig. 6.8 Effect of the enclosure stratification on heat transfer from air to the cold vertical wall (after 50 mins)

- Fig. 6.9 Effect of the enclosure stratification on heat transfer from air to the cold vertical wall (after 60 mins)
- Fig. 6.10 Effect of the enclosure stratification on heat transfer from air to the cold vertical wall (after 80 mins)
- Fig. 6.11 γ versus time
- Fig. 6.12 Nusselt number versus time
- Fig. 6.13 Nusselt number versus Rayleigh number
- Fig. 6.14 Rayleigh number versus time
- Fig. 6.15 Temperature versus time
- Fig. 6.16 Air temperature difference between top most and bottom layer of air at different x distance with time
- Fig. 6.17 Average highest and lowest temperature difference Dt versus time
- Fig. 6.18 Effect of the enclosure on heat transfer from air to the cold vertical wall ($x=0.03m$)
- Fig. 6.19 Temperature distribution of air in the enclosure at steady state condition

LIST OF TABLES

- Table-1 Temperature T ° C at different height from the base and at different distance from the cold plate after 15 minutes
- Table-2 Temperature T ° C at different height from the base and at different distance from the cold plate after 25 minutes
- Table-3 Temperature T ° C at different height from the base and at different distance from the cold plate after 50 minutes
- Table-4 Temperature T ° C at different height from the base and at different distance from the cold plate after 60 minutes
- Table-5 Temperature T ° C at different height from the base and at different distance from the cold plate after 80 minutes
- Table-6 Critical angle for inclined rectangular cavities

CHAPTER-ONE

INTRODUCTION



1.1 Introduction of natural convection and its physical consideration

Natural or free convection is observed as a result of the motion of fluid due to density changes arising from heating or cooling processes. In natural convection flows, the fluid velocities are highly dependent on the thermal field and variable properties can have strong effects on both velocities and temperature. The movement of the fluid in free convection whether it is a gas or a liquid, results from buoyancy forces imposed on the fluid when its density in the proximity of heat transfer surface is decreased or increased as a result of heating or cooling process. Buoyancy forces will not be present if the fluid is not acted upon by some external force fields such as gravity, although gravity is not the only type of force field which can produce the free convection currents. A centrifugal force field can create convection currents. A fluid enclosed in a rotating machine is acted upon by a centrifugal force field if one or more surfaces in contact with the fluid are heated or cooled. Among all properties the temperature gradient has been found to be the most common situation in which a density gradient may arise in a fluid. For gases, density depends on temperature.

Situations for which there is no forced velocity, and yet convection currents exist within the fluid are referred to as free or natural convection. They originate when a body force acts on a fluid in which there are density gradients. The net effect is a buoyancy force, which induces free convection currents. In the most common case, the density gradient is due to a temperature gradient, and the body force is due to the gravitational field. Since free convection flow velocities are generally much smaller than those associated with forced convection, the corresponding convection transfer rates are also smaller. It is perhaps tempting therefore to attach less significance to free convection processes. This temptation

should be resisted. In many systems involving multi-mode heat transfer effects, free convection provides the largest resistance to heat transfer and therefore plays an important role in the design or performance of the system. Moreover, when it is desirable to minimize heat transfer rates or to minimize operating costs, free convection is often preferred to forced convection.

There are, of course, many applications. Free convection strongly influences heat transfer from pipes and transmission lines, as well as from various electronic devices. It is important in transferring heat from electric baseboard heaters or steam radiators to room air and in dissipating heat from the coil of a refrigeration unit to the surrounding air. It is also relevant to the environmental sciences, where it is responsible for oceanic and atmospheric motions, as well as related heat transfer processes.

Physical considerations :

In free convection, fluid motion is due to buoyancy forces within the fluid, while in forced convection it is externally imposed. Buoyancy is due to the combined presence of a fluid density gradient and a body force that is proportional to density. In practice the body force is usually gravitational, although it may be a centrifugal force in rotating fluid machinery or a Coriolis force in atmospheric and oceanic rotational motions. There are several ways in which a mass density gradient may arise in a fluid, but in the most common situation it is due to the presence of a temperature gradient. We know that the density of gases and liquids depends on temperature, generally decreasing (due to fluid expansion) with increasing temperature ($\delta\rho / \delta T < 0$).

In this topic we focus on free convection problems in which the density gradient is due to temperature gradient and the body force is gravitational. However, the presence of a fluid density gradient in a gravitational field does not ensure the existence of free convection currents. Consider the conditions of Figure 1.1. A fluid is enclosed by two large, horizontal plates at different temperatures ($T_1 \neq T_2$). In case (a) the temperature of the lower plate

exceeds that of the upper plate, and the density decreases in the direction of the gravitational force. This condition is unstable, and free convection currents will exist. The gravitational force on the denser fluid in the upper layers exceeds that acting on the lighter fluid in the lower layers, and the designated circulation flow pattern will commence. The heavier fluid will descend, being warmed in the process, the lighter fluid will rise and will be cooled by descending cold fluid. However, this condition does not characterize case (b), for which $T_1 > T_2$ and the density no longer decreases in the direction of the gravitational force. Conditions are now stable, and there is no bulk fluid motion. In case (a) heat transfer occurs from the bottom to the top surface by free convection; for case (b) heat transfer (from top to bottom) occurs by conduction.

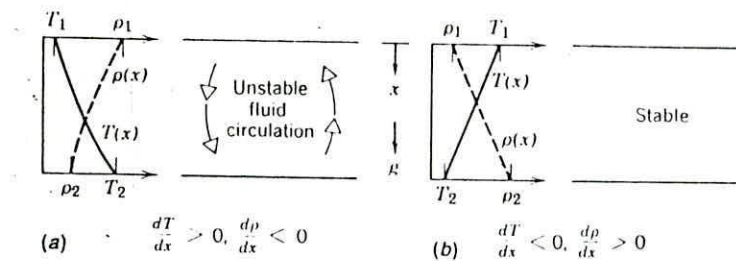
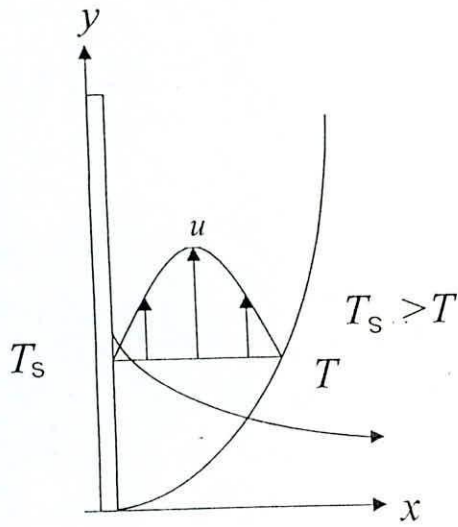
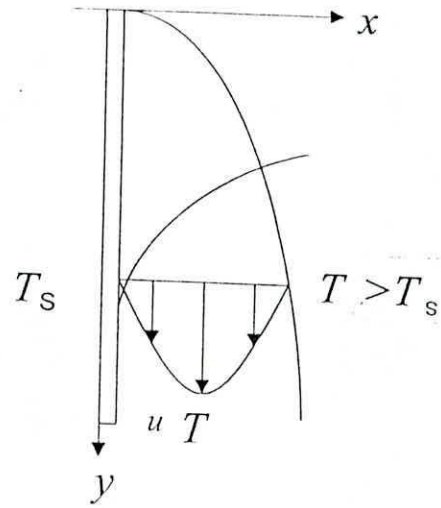


Fig. 1.1: Conditions in a fluid between large horizontal plates at different temperatures (a) Unstable temperature gradient (b) Stable temperature gradient.

In this text we focus on free convection flows bounded by a surface, and a classical example relates to boundary layer development on a heated vertical plate (Figure 1.2a). The plate is immersed in an extensive, quiescent fluid, and with $T_s > T$, the density of fluid close to the plate is less than that of fluid at a distance from the plate. Buoyancy forces therefore induce a free convection boundary layer in which the heated fluid rises vertically, entraining fluid from the quiescent region. The resulting velocity distribution is unlike that associated with forced convection boundary layers. In particular the velocity is zero as $y \rightarrow \infty$, as well as at $y = 0$. A free convection boundary layer also develops if $T_s < T$. In this case, however, fluid motion is downward (Fig. 1.2b).



a



b

Fig. 1.2a: Boundary layer development on a heated vertical plate.

Fig. 1.2b: Boundary layer development on a cooled vertical plate.

1.2 Natural convection in an enclosure

In many engineering applications natural convection plays an important role as a dominating mechanism of heat transfer. Heat transfer by free convection in enclosed spaces has numerous engineering applications. For example, double glazing, nuclear insulation, ventilation room, solar energy collector, crystal growth in liquid, the periodic energizing of electronic devices by the on and off heating and cooling mode, effective cooling of microelectronic equipment, refrigeration and air conditioning system with chilled ceiling etc. There are two elementary classes of natural convection flows in enclosure, namely, those in vertical enclosures with two differentially heated vertical walls and those in horizontal enclosures with two differentially heated horizontal walls. The enclosure may be horizontal, vertical or inclined. The determination of the on-set of free convection and the heat transfer coefficient in an enclosure associated with free convection has been the subject of numerous

investigations. Despite the vast number of experimental and analytical studies of free convection in enclosed spaces, the heat transfer correlation covering all ranges of parameters are not available. Numerous studies on natural convection in enclosure related to either side heating or bottom heating have been reported. But the natural convection in enclosure with one side cold and other sides adiabatic is relatively unknown. The natural convection heat transfer depends on boundary layer thickness. Hence the development of the flow is often strongly influenced by the shape of the boundaries. Therefore the heat transfer rate and heat transfer coefficient are also dependent on the boundary conditions of enclosure.

1.3 Effect of thermal stratification

Natural convection along a vertical cold wall is the simplest model possible, heat transfer interaction between a vertical wall and an isothermal semi infinite fluid reservoir is shown in Fig. 1.3. We take a closer look at the problems of modeling a heat transfer situation involving natural convection. Vertical walls are rarely in communication with semi infinite isothermal pools of fluid. More often their height is finite and the cold boundary layer eventually hits the bottom. At that point the cold stream has no choice but to discharge horizontally into the fluid reservoir. The direction of this discharge is horizontal because the discharge contains fluid colder than the rest of the reservoir. The long time effect of the discharge process is the thermal stratification characterized by warm fluid layers floating on top of colder layers. At this point it is sufficient to recognize that the air in any room with the door closed is thermally stratified in such a way that the lower layers assume the temperature of the coldest wall and the highest layer near the ceiling approach the temperature of the warmest wall.

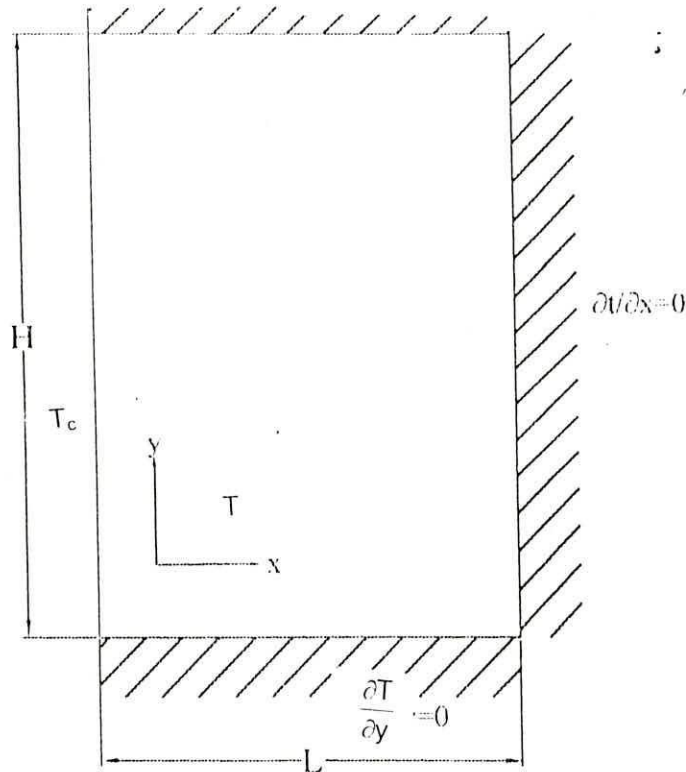


Fig. 1.3: Natural convection along a cold vertical wall environment.

1.4 Motivation behind the selection of present work

In many engineering applications natural convection plays an important role as a dominating mechanism of heat transfer [1]. The phenomenon of natural convection in fluid-filled enclosure has received considerable attention in recent years. This is mainly because of the this fact that phenomenon substantially influence the thermal performance of heat transfer equipment used for cooling of electronic components, refrigeration and air-conditioning system etc. [2-4]. Many studies have been done regarding natural convection in an enclosure. Most of the studies on natural convection in an enclosure are related to either vertical wall heating or bottom surface heating [5-8]. The present work deals with the natural convection in rectangular enclosure where one of its side walls will be cooled and the top and bottom surfaces will be adiabatic.

1.5 The Present study

An experimental investigation on the natural convection heat transfer in a rectangular enclosure from one cooled side wall is conducted with the following objectives:

- i. To measure temperatures of the cold vertical wall and air at different locations within the enclosure during transient period .
- ii. To measure temperatures of the cold vertical wall and of air at different locations within the enclosure during steady state condition.
- iii. To study the variation of average natural convection heat transfer coefficient h at different thermal potentials, ΔT .
- iv. To understand thermal stratification phenomenon for this enclosure.
- v. To determine the Rayleigh number at different locations within the enclosure for both transient and steady state conditions.
- vi. To develop a correlation which will be recommended for estimation of heat transfer coefficient for natural convection in the rectangular enclosure cooled from one of its vertical walls.

CHAPTER-TWO

LITERATURE REVIEW

2.1 Introduction

Fluid motions and transport processes due to buoyancy effect plays an important role in various field of science and technology. As a result, this subject is currently discussed in conferences and journals covering such diverse areas of meteorology, geophysics, astrophysics, nuclear reactor systems, materials processing, solar energy systems, energy storage and conservation fire control, and chemical, food, and metallurgical industries, as well as in the more conventional fields of fluid and thermal sciences.

Buoyancy-induced flows are complex because of the essential coupling between the flow and transport. The problems can be classified as either external ones (free convection) or internal ones (natural convection). The first unified and comprehensive review of this subject was made by Ostrach[9]. Later, summaries of free convection were presented by Ede[10] and Gebhart[11] and other reviews of natural convection were compiled by Ostrach[12], Catton[13], Ostrach[14] and Hogendoorn[15]. Each of the last three emphasize essentially on different aspects of the subject.

It was first pointed out by Ostrach[16] that internal problems are considerably more complex than the external ones. This is because at large Rayleigh numbers (the product of Prandtl and Grashof numbers) classical boundary layer theory yields the same simplifications for external problems that are so helpful in other fluid-flow problems, viz., the region exterior to the boundary layer is unaffected by the boundary layer. For confined natural convection, on the other hand, boundary layer forms near the walls but the region exterior to them is enclosed by the boundary layers and forms a core region. Since the core is partially or fully encircled by the boundary layers, the core flow is not readily determined from the boundary conditions but

depends on the boundary layers, which, in turn, is influenced by the core. The interactions between the boundary layer and the core constitute a central problem that has remained unsolved and is inherent to all confined convection configurations, namely, the flow pattern cannot be predicted a priori from the given boundary conditions and geometry. In fact, the situation is even more intricate because it often appears that more than one global core flow is possible and flow subregions, such as cells and layers, may be embedded in the core. This matter, which has been discussed more fully by Ostrach[12-14], and Ostrach and Hantman[17], is not merely a subtlety for analysis, but has equal significance for numerical and experimental studies, as will be indicated later.

It is distressing to note that this crucial aspect of natural convection seems to be essentially ignored in most existing literature or is treated in a cavalier manner. The core flow is often merely assumed, estimated in an adhoc manner, or specified from seemingly similar problems. However, experience has shown that natural convection is extremely sensitive to changes in the container configuration and the imposed boundary conditions so the use of results from similar problems is dangerous. In numerical studies, the entire matter generally is ignored and, as a consequence, there have been no truly reliable predictive result of velocity and temperature distributions. Until the problem is resolved, numerical studies must be guided and closely coupled with experiments.

Lack of progress on this crucial aspect of natural convection can be attributed to two reasons, one old and one relatively new. The first is related to proper normalization of equations, sometimes referred to as dimensional or scaling analysis. As was pointed out by Ostrach, it is disturbing to note that despite the existence of clear and explicit delineations of how to normalize natural-convection problems, they are scaled in many different inappropriate ways (even for identical problems) in the literature. This can lead to errors in analysis and to considerable numerical problems (see de Vahl Davis, 1986), some of which even lead to misrepresentation of the physics. Scaling analysis is also essential or indicating the resolution required in both numerical and experimental studies. The disregard or lack of appreciation of such vital aspects, which are well documented, is hard to understand. The second aspect

alluded to above concerns inherent coupling between the flow and the buoyancy-driving force. This was first indicated by Hantman[18] (1969) and later noted more specifically by Cormack et al. [19], but was not fully appreciated until relatively recently. In essence, this coupling causes the principal driving force to act in different regions of the enclosure, depending on conditions. This unusual physical characteristic of natural convection must be properly accounted for if meaningful results are to be obtained.

To add to all the complexity it should be recalled that there are essentially two basic modes of flow generated by buoyancy. The first, usually referred to as conventional convection, occurs whenever a density gradient (due to thermal and/or concentration effects) is normal to the gravity vector. In such a case a flow ensues immediately. The second mode, called unstable convection, occurs when the density gradient is parallel but opposed to the gravity vector. In this situation the fluid remains in a state of unstable equilibrium (due to heavier fluid being above the lighter) until a critical density gradient is exceeded. A spontaneous flow then results that eventually becomes steady and cellular-like. If the density gradient is parallel but in the same direction as gravity the fluid is stably stratified. As if all this were not sufficiently difficult to deal with, both conventional and unstable (or stratified) modes can interact in a given configuration.

For many years research on natural convection in enclosures centered around two basic configurations, viz., horizontal cylinders and rectangular cavities. The height-to-width (aspect) ratio of the rectangular cavities ranged from values equal to or greater than unity. Most attention has been given to each mode of convection separately, i.e., where the density gradients in a gravitational field were either horizontal or vertical with the gradient increasing upward. Most density variations were considered to be the result of temperature gradients. Less attention has been given to situations in which the buoyancy is due to concentration differences. Catton[13] reviewed work done when both modes occurred simultaneously, as, in tilted rectangular enclosure.

More recently, due to motivations from diverse applications, the research scope has expanded in many ways. The configurations now being considered are low-aspect-ratio rectangles, parallelograms, annuli, and three-dimensional enclosures. The fluids now may be radiation participating media, or be changing phase, or be in porous media. Some problems are now being considered with heat flow in more than a single direction, either by direct imposition or by inclination of the cavity. Problems are being considered in which the imposed thermal conditions are not specified but must be determined from the interaction between the enclosure heat transfer and the environment or cover only part of the surface. Increasing attention is being given to situations in which the buoyancy is due to the combined effects of temperature and concentration gradients, which have various orientations relative to themselves and to gravity.

Clearly, with the limited space allotted to this paper, it is impossible to present a comprehensive review of all ongoing research. Because many physical and mathematical questions remain unresolved and others are introduced by some new works, emphasis will be given to refocusing on the crucial aspects of the conventional convection problem and on presenting insights on how to deal with them. In this way, some other older work will be revisited and some of the most recent work will be discussed.

2.2 Overview of Early Work

The early work done on fully developed flows in channels and tubes, closed-end tubes, and in a shallow right circular cylinder that rotate about its axis and is heated at its lower surface was reviewed by Ostrach[11-12].

The earliest investigation of natural convection in a completely confined configuration seems to have been made by Lewis[20], who considered the heat transfer of foam type commercial insulating materials, which consist of gas filled cells dispersed throughout a solid material. The natural convection in the cells was of particular interest. To simplify the analysis a spherical cell was treated as a horizontal cylinder of circular cross section and infinite length.

A cosine wall temperature was specified, with the maximum and minimum wall temperatures on opposite ends of the horizontal diameter. This temperature corresponds to that which would occur in the solid without gas bubbles. Since the cells were very small, Lewis considered only the case of Rayleigh number less than unity for which no boundary-layer phenomena are encountered. A similar problem was treated by Zhukovitskiy[21]. An excellent survey of low Rayleigh number problems is presented by Ostroumov[22]. Low Rayleigh number convection in a spherical cavity was investigated by Drakhin[23].

The heat transfer through gas layer confined in rectangular cavities composed of isothermal vertical walls and either perfectly conducting or perfectly insulating horizontal surfaces was investigated by Batchelor[24] for the cases of height-to-width ratios of layers between 5 and 200. It was reasoned that several different flow regimes could occur within the cavity depending on the values of the height-to-width ratio l/d and the Rayleigh number Ra . For small Ra Batchelor[24] used a perturbation scheme similar to that of Lewis[25] and, as expected, concluded that convection was unimportant compared to conduction. Conduction was also found to be the sole means of heat transfer in the asymptotic case of $l/d \propto Ra$ and general Ra . In such cases, the temperature distribution in the air was found to vary linearly between the walls and fluid flow was entirely vertical. Conduction was restricted to the upper and lower ends of the cavity and as Ra approached values appropriate for boundary layers, these effects propagated into the rest of the cavity. This asymptotic case of infinite aspect ratio had been investigated earlier by Ostrach[26] who treated it as a special case in a general analysis of natural convection between parallel and vertical isothermal plate.

None of the aforementioned difficulties were encountered in those problems because the flows were not of the boundary layer type.

2.3 Empirical Correlations: Enclosures

Many engineering applications frequently involve heat transfer surfaces that are at different temperatures and are separated by an enclosed fluid. Now we present correlations that are pertinent to the most common geometries.

2.3.1 Rectangular Cavities

The rectangular cavity, shown schematically in Fig. 2.1, has been studied extensively and comprehensive reviews of both experimental and theoretical results are available in the literature [27, 28]. The tilt angle τ between the heated and cooled surfaces of the cavity and the horizontal plane strongly influences the heat flux across the cavity. This flux may be expressed as

$$q'' = h (T_1 - T_2) \quad \dots\dots 2.1$$

The horizontal cavity heated from below ($\tau = 0$) has been considered by many investigators. It is known that when

$$Ra \equiv \frac{g\beta(T_1 - T_2)L^3}{\alpha\nu} > 1708$$

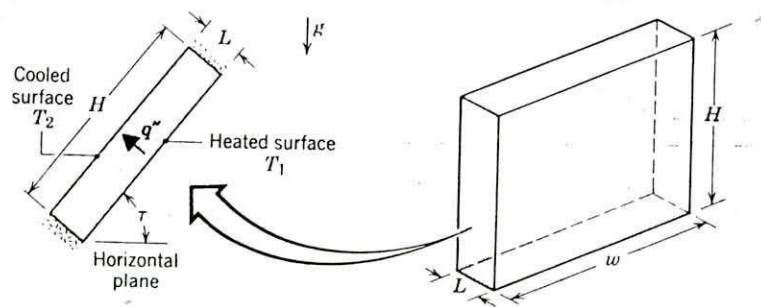


Fig. 2.1: Free convection in a rectangular cavity.

conditions are thermally unstable and there is heat transfer by free convection. As a first approximation, the convection coefficient may be obtained from a correlation proposed by Globe and Dropkin [29].

$$\bar{Nu}_L = \frac{\bar{h}L}{k} = 0.069Ra_L^{1/3} Pr^{0.074} \quad 3 \times 10^5 < Ra_L < 7 \times 10^9 \quad \dots\dots 2.2$$

where all properties are evaluated at the average temperature, $\bar{T} \equiv (T_1 + T_2)/2$. More detailed correlations, which apply over a wider range of Ra_L , have been proposed [30-31]. In each case, however, the ratios L/H and L/w were small, and the effects of lateral boundaries on heat transfer were negligible. For $Ra_L < 1708$ or for $\tau = 180^\circ$, heat transfer between the surfaces occurs by conduction and $Nu_L = 1$.

In the vertical rectangular cavity ($\tau = 90^\circ$) the vertical surfaces are heated and cooled, whereas the horizontal surfaces are adiabatic. For aspect ratios in the range $1 < (H/L) < 10$, the following correlations have been suggested [28].

$$\bar{Nu}_L = 0.22 \left(\frac{Pr}{0.2 + Pr} Ra_L \right)^{0.28} \left(\frac{H}{L} \right)^{-1/4} \quad \dots\dots 2.3$$

$$\left[\begin{array}{l} 2 < (H/L) < 10 \\ Pr < 10^5 \\ Ra_L < 10^{10} \end{array} \right]$$

$$\bar{Nu}_L = 0.18 \left(\frac{Pr}{0.2 + Pr} Ra_L \right)^{0.29} \quad \dots\dots 2.4$$

$$\left[\begin{array}{l} 1 < (H/L) < 2 \\ 10^{-3} < Pr < 10^{-5} \\ 10^{-3} > (Ra_L Pr)/(0.2 + Pr) \end{array} \right]$$

For larger aspect ratios, the following correlations have been proposed [27]:

$$\bar{Nu}_L = 0.42 Ra_L^{1/4} Pr^{0.012(H/L)^{-0.3}} \begin{cases} 10 < (H/L) < 40 \\ 1 < Pr < 2 \times 10^4 \\ 10^4 > Ra_L < 10^7 \end{cases} \quad \dots 2.5$$

$$\bar{Nu}_L = 0.46 Ra_L^{1/3} \begin{cases} 1 < (H/L) < 40 \\ 1 < Pr < 20 \\ 10^6 > Ra_L < 10^9 \end{cases} \quad \dots 2.6$$

Convection coefficients computed from the foregoing expressions are to be used with Equation 2.1. Again, all properties are evaluated at the mean temperature, $(T_1 + T_2)/2$.

Many publications have appeared in recent years concerning free convection heat transfer in inclined, rectangular spaces, with particular attention given to solar collector applications. For large aspect ratios, $(H/L) > 12$, and tilt angles less than the critical value τ^* given in Table 6 the following correlation due to Hollands et al. [33] is in excellent agreement with available data.

$$\bar{Nu}_L = 1 + 1.44 \left[1 - \frac{1708}{Ra_L \cos \tau} \right]^* \left[1 - \frac{1708 (\sin 1.8\tau)^{1.6}}{Ra_L \cos \tau} \right] + \left[\left(\frac{Ra_L \cos \tau}{5830} \right)^{1/3} - 1 \right]^* \begin{cases} (H/L) > 12 \\ 0 < \tau \leq \tau^* \end{cases} \quad \dots 2.7$$

The notation []* implies that, if the quantity in brackets is negative, it must be set equal to zero. For small aspect ratios Catton [28] suggests that reasonable results may be obtained from a correlation of the form

$$\bar{Nu}_L = \bar{Nu}_L(\tau = 0) \left[\frac{\bar{Nu}_L(\tau = 90)}{\bar{Nu}_L(\tau = 0)} \right]^{\tau/\tau^*} (\sin \tau^*)^{(\tau/4\tau^*)} \begin{cases} (H/L) < 12 \\ 0 < \tau \leq \tau^* \end{cases} \quad \dots 2.8$$

Beyond the critical tilt angle, the following correlations due to Ayyaswamy and Cartton and Arnold et al. [31], respectively , have been recommended [28] for all aspect ratios (H/L).

$$\bar{Nu}_L = 1 + \bar{Nu}_L(\tau = 90^\circ)(\sin \tau)^{1/4} \quad \tau^* < \tau < 90^\circ \quad \dots\dots 2.9$$

$$\bar{Nu}_L = 1 + [\bar{Nu}_L(\tau = 90^\circ) - 1]\sin \tau \quad 90^\circ < \tau < 180^\circ \quad \dots\dots 2.10$$

CHAPTER-THREE

MATHEMATICAL MODELING OF THE PROBLEM

3.1 Description of the Problem

The natural convection heat transfer in a rectangular enclosure is a complex problem of temperature, boundary conditions, position of the enclosure and the properties of the confined fluid. The rectangular enclosure of the present investigation has a cold side wall. The other side walls, bottom surface and ceiling are adiabatic as shown Fig. 3.1. The convective fluid is air. The flow, temperature and concentration fields are taken as two dimensional. At time less than or equal to zero, the fluid in the enclosure is quiescent and the temperature is uniform. For time $t > 0$ cold plate absorbs heat from the air. The fluid comes in contact with the cold plate, the temperature of the cold plate is gradually decreasing and finally reaches a steady state value. During transient and steady state condition the heat flux q'' was transferred to the cold plate by convection.

3.2 Governing Equation

The geometry considered is a two-dimensional air layer confined by two horizontal impermeable and adiabatic rigid walls and two vertical walls. The air of the enclosure is initially at rest and isothermal at room temperature T_α at which the density of air is minimum. The back side is cooled gradually in lower temperature.

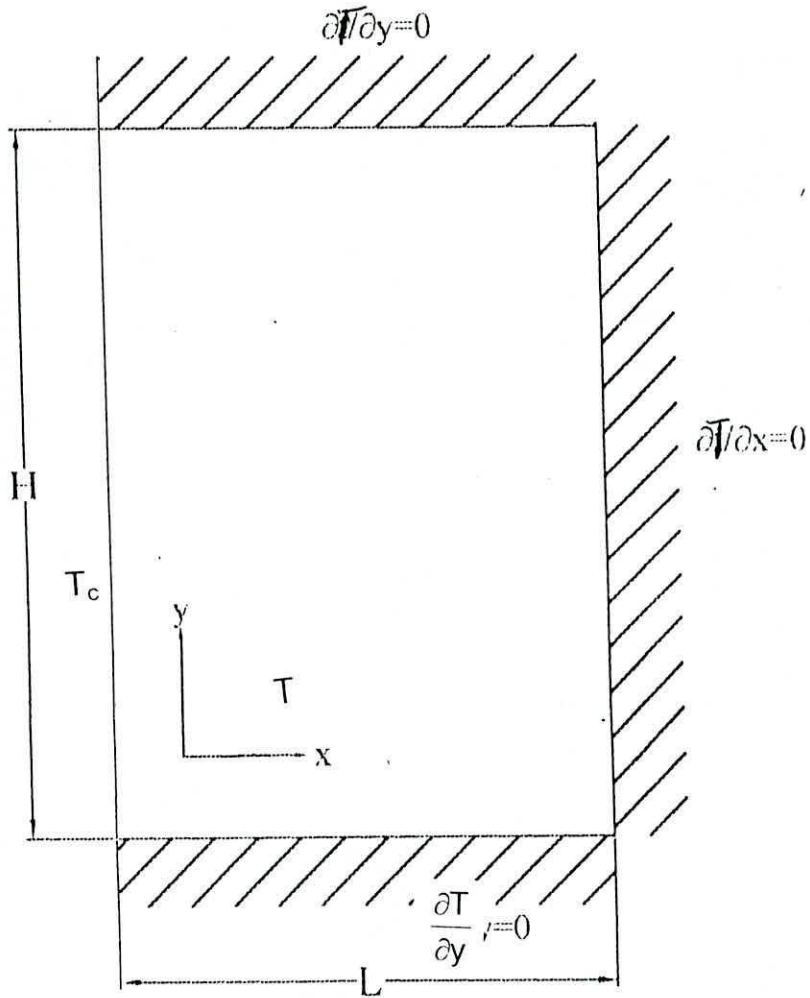


Fig. 3.1: Schematic configuration of the problem and coordinate system.

For measuring temperature of air in the enclosure at different locations, twelve thermocouples are placed in three columns and four rows, thus, four vertical air sub-layers are formed with various density gradients. The flow is assumed to be Newtonian, two dimensional buoyancy induced flow with negligible viscous dissipation. The governing equations are the continuity, momentum and energy equations:

$$\nabla \cdot \mathbf{u} = 0 \quad \text{----- (1)}$$

$$\frac{\partial \mathbf{u}}{\partial t} + \mathbf{u} \cdot \nabla \mathbf{u} = -\frac{1}{\rho_o} \nabla p + \nu \nabla^2 \mathbf{u} + \frac{\rho}{\rho_o} \mathbf{g} \quad \text{----- (2)}$$

$$\frac{\partial T}{\partial t} + \mathbf{u} \cdot \nabla T = k \nabla^2 T \quad \text{----- (3)}$$

where u is the velocity vector, ν the kinetic viscosity, T the temperature, p the pressure, t the time, ρ_0 the minimum density at T_α and g is the acceleration due to gravity.

There are several models for predicting the density-temperature behavior of air around the maximum density region. A parabolic density temperature relationship is given by Debler[34].

$$\frac{\rho}{\rho_0} = 1.0 - \nu(T_0 - T)^2 \quad \text{-----(4)}$$

Later, by adding cubic term in equation (4) this model can be expanded to decrease the temperature in the range 27 and -7°C .

$$\frac{\rho}{\rho_0} = 1.0 - \nu_1(T_0 - T)^2 + \nu_2(T_0 - T)^3 \quad \text{-----(5)}$$

The non-slip boundary condition is imposed at all rigid walls. The initial and boundary conditions are specified as follows:

(i) for time $T < 0$
 $u = 0; T = T_0 \forall x, \forall y \quad \text{----- (6)}$

(ii) for $T > 0$
 $u = 0; T = T_0$ at $x = 0$
 $u = 0; T = T_h$ at $x = L$
 $u = 0; \frac{\partial T}{\partial y} = 0$ at $y = 0; y = H. \quad \text{-----(7)}$

The dimensionless variables (geometry, velocity, temperature, pressure and time) are introduced as follows:

$$\begin{aligned}
X &= \frac{x}{L} & Y &= \frac{y}{L} \\
U &= \frac{u}{\frac{k}{L} \sqrt{(Ra Pr)}}; \theta = \frac{T_o - T}{T_h - T_c} \\
P &= \frac{p}{\frac{\mu k}{L^2} \sqrt{(Ra Pr)}}; t = \frac{T}{\frac{L^2}{k} \sqrt{(Ra Pr)}}
\end{aligned}
\tag{8}$$

where L is the enclosure length, k the thermal diffusivity, and μ the dynamic viscosity. The Rayleigh number, Ra , and the Prandtl number, Pr , are defined as

$$Ra = \frac{g\gamma(T - T_c)L^3}{kv} \tag{9}$$

$$Pr = \frac{\nu}{k} \tag{10}$$

Thus, with the definitions in (8) the dimensionless governing equations expressing conservation of mass, momentum, and energy are written as:

$$\nabla \cdot \mathbf{U} = 0 \tag{11}$$

$$\sqrt{\frac{Ra}{Pr}} \left(\frac{\partial \mathbf{U}}{\partial t} + \mathbf{U} \cdot \nabla \cdot \mathbf{U} \right) = -\nabla P + \nabla^2 \mathbf{U} - \sqrt{\frac{Ra}{Pr}} \theta^2 \tag{12}$$

$$\sqrt{(Ra Pr)} \left(\frac{\partial \theta}{\partial t} + \mathbf{U} \cdot \nabla \cdot \theta \right) = \nabla^2 \theta \tag{13}$$

From equation (12) it can be seen that the buoyancy effect is characterized by the ratio of two dimensionless parameters: Ra and Pr . In this study, Pr is kept fixed at 0.7 .

The heat transfer rate at the vertical walls is described by the Nusselt number, which is a function of both time and space. In terms of dimensionless parameters the local Nusselt number is defined as:

$$Nu = \left. \frac{\partial \theta}{\partial X} \right|_{x=0.1} \quad \text{----- (14)}$$

The average Nusselt number is the integral of the local Nusselt number over the length of the vertical wall

$$\bar{Nu} = \frac{1}{A} \int_0^H \left. \frac{\partial \theta}{\partial X} \right|_{x=0.1} dY \quad \text{-----(15)}$$

3.3 Mathematical Equations

A two dimensional (2D), steady state and transient incompressible laminar flow model is considered in the present study. All properties are assumed to be constant except the effect of density.

The mathematical equations that are used to calculate the natural convection heat transfer coefficient, Nusselt number, and Rayleigh number are given below:

$$Q = -k_f A \frac{\partial T}{\partial x} = hA(T_\alpha - T_c)$$

$$Nu_H = h_t L / K$$

$$Ra_H = g\beta\Delta T l^3 / \nu^2 \times Pr$$

$$\gamma = \frac{\Delta T}{H}$$

CHAPTER-FOUR

EXPERIMENTAL SETUP

4.1 General Description of the Experimental Facility

This chapter describes the experimental aspects of the investigation and includes a detailed description of the experimental facility. Fig. 4.1 presents schematically the experimental set up and test section respectively. Major components of test apparatus are, rectangular enclosure, thermoelectric module, fan, rectifier converter heat sink, cold plate and data acquisition system.

The dimensions of the rectangular enclosure are: base 40 cm x 35 cm and height 50 cm. Internal space of the rectangular enclosure is: base 32cm x 32cm and height 45cm. The enclosure is made of mild steel sheet. Each of the walls is thermally insulated with styrofoam. A thermoelectric module of size 9x4x7cm is placed at the center of one of the vertical walls of the enclosure. An aluminum plate of size 22cm x 22cm and of thickness 5mm is screwed to the thermoelectric module. In course of the experiment, cooling of the aluminum plate is accomplished by the thermoelectric module (TEM). In first setting, 14 thermocouples are used out of which 12 thermocouples, are used to measure the temperature of air within the enclosure at different locations along and from the cold plate (Fig. 4.2). The other two thermocouples are used to measure cold plate temperature and room temperature. The input current is measured by a precision ammeter. Forced convection heat sink (fins) of area 1080 cm² are used on the hot side of the heat exchanger to give off the heat to the surrounding air. For forced convection 220 volt (AC), 50hz, ¼ hp fan is used over the sink. For supplying 10 volt (DC) to the thermoelectric module a rectifier is used. A data acquisition system is used to record the data from the thermocouples.

4.2 Test Section

The test section consist of :

- i. Rectangular enclosure
- ii. Thermoelectric module
- iii. Data acquisition system
- iv. Fan
- v. Fins
- vi. Rectifier
- vii. Thermocouples

Details of the test section is shown in the Fig. 4.1

4.2.1 Rectangular enclosure

All walls of the enclosure are made of mild steel. The thickness of the mild steel sheet is 1mm. All walls are thermally insulated with styrofoam. The thickness of the styrofoam is 2.54cm. There is a door on the front side of the enclosure. The provision of the door is kept for placing the thermocouple at different locations in the enclosure and also for screwing up the aluminum plate with the thermoelectric module. The thermoelectric module is placed at the centre of one of the vertical walls. Heat sink (fin) is used on the hot side of the heat exchanger to give off the heat to the surrounding air. For force a convection low capacity fan is used over the sink Fig. 4.1.

4.2.2 Thermoelectric module

For cooling the internal space of the enclosure a thermoelectric module is used (Fig. 4.3). The dimension of thermoelectric module is of 9cm x 4cm and height 7cm. The thermoelectric module has 11 couples. The module is composed of two ceramic substrates that serve as a foundation and electric insulation for P-type and N-type bismuth telluride dice that are

connected electrically in series and thermally parallel between the ceramics. The ceramics also serve as insulation between modules, internal electric elements and a heat sink that must be in contact with the hot side as an object against the cold side surface. An electrically conductive material, copper pads attached to the ceramics, maintain electric connections inside the module. Solder is used at the joints to enhance the electric connection and hold the module together.

Most modules have even number of P-type and N-type dice and one of each sharing has an electrical interconnection and is known as, "a couple." The above module would be described as an 11-couple module.

While both P-type and N-type materials are alloys of bismuth and tellurium, both have different free electrical densities at the same temperature. P-type dice are composed of material having a deficiency of electrons and N-type has an excess of electrons. As current (amperage) flows up and down through the module it attempts to establish a new equilibrium within the materials. The current treats the P-type material as a hot junction needing to be cooled and the N-type as a cold junction needing to be heated. Since the material is actually at the same temperature, the result is that hot side becomes hotter while the cold side becomes colder. The direct current will determine if a particular die will cool down or heat up.

4.2.3 Data acquisition system

The system consist of (1) data acquisitor (2) UPS (3) CPU (4) Monitor

Data acquisitor

The data acquisitor used to measure temperature at different locations is the component number 2 as shown in Fig. 4.1. The acquisitor model is COLE-PARMER PCA-14. From the

certifications of construction the resolution and accuracy are found to be 16 bits and $\pm 0.02\%$ respectively. The dimension is height 35cm, width 10cm and depth 22.5 cm.

The software supplied is menu driven and very easy to use with BASIC interpreter supplied with any computer. The menu appears on the monitor and enables to identify each channel, set up the name of data file, vary the time between storing sets of data on disk, display disk directories etc. No knowledge of programming is needed.

4.2.4 Rectifier

To supply the DC power to the thermoelectric module and data aqisitor two rectifiers are used. The input and output of the rectifiers are 200-250v A.C ,50 Hz and 10v D.C,3A respectively . Out put voltage variation is $\pm 0.1 \%$ (max).

4.2.5 Thermocouples

T type thermocouple are employed in this experiment.

a. Position of the thermocouples in the enclosure:

To measure the temperature of air within the enclosure at different height and different locations along and from the cold plate 12 thermocouples are used as shown in Fig. 4.2. Thermocouples are placed in three columns and four rows. The position of three columns are 30mm, 150mm and 300mm apart from the cold plate and the position of rows are 20mm, 150mm 300 mm and 400 mm apart from the bottom surface of the enclosure. Two thermocouples are used to measure cold plate temperature and room temperature.

4.3 Details of the cold plate assembly

To receive heat from air in the enclosure cold plate assembly is placed on the back vertical wall of the enclosure .To make all walls adiabatic, 2.54 cm thick styrofoam is used. A

rectangular hole of size 9 x 4 cm is made at the middle of the back vertical wall of the enclosure . Then a thermoelectric module of 9cm x 4cm and height 7cm is placed in the rectangular hole. An aluminium (cold) plate of size 23cm x 25cm and of thickness 5mm is screwed to the inside (cold side) of the thermoelectric module. Heat sink (fins) about 1080-square cm is used on the outside (hot side) of the thermoelectric module to give off the heat to the surrounding air. For forced convection a fan is placed against the sink (fig. 4.4).

CHAPTER-FIVE

EXPERIMENTAL MEASUREMENT AND TEST PROCEDURE

5.1 Instrumentation

In the present investigation provision are made for measuring temperature of air within the enclosure at different locations and the temperature of the cold plate. AC to DC converter is used to supply the necessary voltage at different sections of the experimental set up. A data acquisition system is used to record the data from thermocouples. Details of the instrumentation and measurement procedure are as follows.

5.2 Calibration of Thermocouple

Calibration of one of the thermocouples is done with the help of a J-type thermocouple. At first, leads of both the thermocouples are connected with respective meters. The junctions of both thermocouples are put into a bowl containing tap water. At the steady state condition, both the meters read 27°C temperature which are recorded. Then some ice chips is mixed with the water of the bowl. After stirring for some time, both the thermocouple meters become static and their readings are recorded. Again, some more ice chips are added to the water until a steady temperature of -2°C is reached. The calibration curve is shown in Fig. 5.1.

5.3 Temperature Measurement

Fourteen 36 SWG T-type thermocouples are used in this experiment. Twelve thermocouples are placed at different locations within the enclosure [Fig. (4.1)] to measure the temperature of air. Other two thermocouples are used to measure the room and cold plate temperature. Data acquisition system is used to record the data from the thermocouples at every five-

minute interval until steady state condition is reached. Recorded data is stored in the computer.

5.4 Current and Voltage Measurement.

Power is measured indirectly by using a digital volt meter and a precision ammeter. The digital volt meter of model LEADER LDM-853A is employed to measure the voltage across the thermoelectric module which absorbs the necessary heat to from the air within the enclosure. The volt meter has the accuracy of $\pm 0.3\%$ and the range of the voltmeter is 0.2 – 1000 v.

A precision ammeter of model WESTING HOUSE 936234E is used to measure the current supplied to the experimental thermoelectric module. It has the sensitivity of ± 0.01 amp.

5.5 Experimental Procedure and Data Analysis

A rectangular enclosure having a height of 450 mm and length 320 mm is used. The experimental fluid is air at atmospheric pressure. The experiment is carried out under transient as well as steady state conditions.

The overall test procedure is as follows:

1. First one end of the fourteen thermocouples are placed at different locations in the enclosure. Other ends of the thermocouples are connected to the aqisitor.
2. Then aqisitor is connected to the CPU of the computer.
3. Electric power is supplied to the aqisitor through a rectifier.
4. Electric power is also supplied to the thermoelectric module through another rectifier.

5. The computer is then started and PCA-14 is run.
6. From main menu fourteen channels are selected.
7. Then data are recorded till steady state condition is reached.
8. Then recorded data are analyzed for finding out heat transfer co-efficient, Nusselt number and Rayleigh number.

CHAPTER-SIX

RESULT AND DISCUSSION

6.1 Discussion on the Results and Finding

Figures 6.1 to 6.5 show the temperature distribution of the air within the enclosure. The data are presented at different height 'Y' in the enclosure at different distance 'X' from the cold plate at different interval. The data at the interval of 15, 25, 50, 60 and 80 minutes are analyzed. From the temperature distribution profile in the enclosure the heat transfer rate Q, heat transfer coefficient "h" and average Nusselt number are calculated.

For finding out heat transfer rate Q temperature data and distance are plotted in sigma plot-2000, and curve fitting is done. It is found that the single rectangular hyperbola fitted the data very well and the nature of the equation can be given as:

$$T = T_c + ax / (b + x)$$

Where

$$T_c = \text{cold wall temperature}$$

$$T = \text{Air temperature of the enclosure}$$

From the above equation dT/dx is calculated for each curve and the average dT/dx was also calculated for each interval and Q is calculated by using the following equation

$$Q = -k_f A dT/dx = hA (T - T_c)$$

Where Q is the total heat transfer from the air to the cold plate during that time.

It is found that the value of heat transfer rate Q increases with the interval of time.

For calculating temperature gradient γ , a general model for gravitational force driven flow near the cold vertical wall is considered as shown in Fig. 6.0. Now the fluid of enclosure is linearly stratified.

$$T(y) = T_0 + \gamma y$$

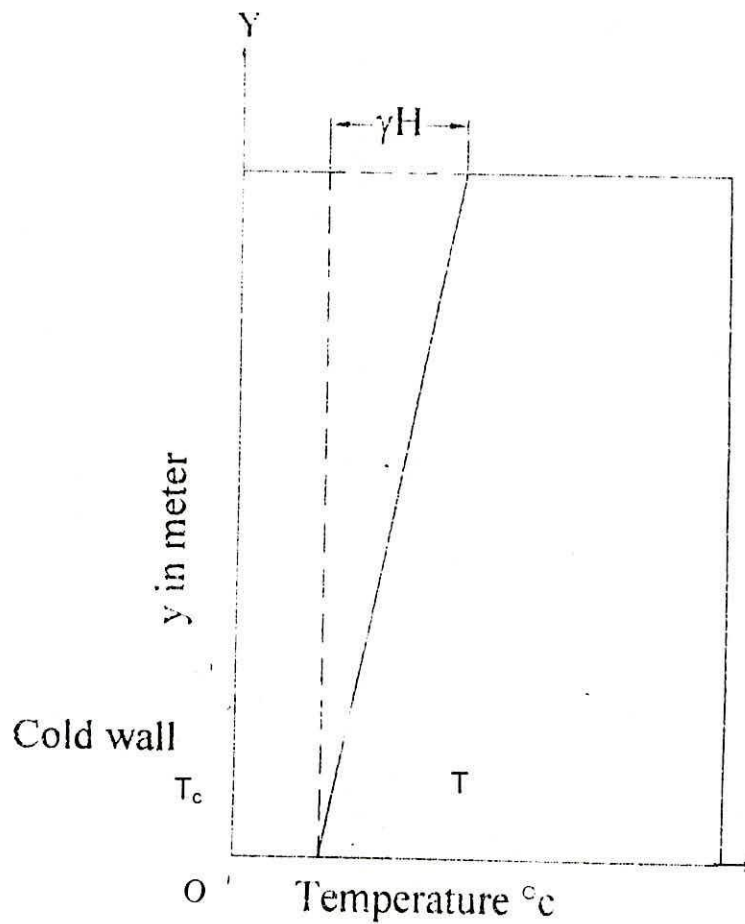


Fig. 6.0: The effect of stratification in a cold vertical wall enclosure.

T.O being the lowest temperature in the arrangement and γ is constant temperature gradient. The dashed line shown in the figure the location of the isothermal reservoir model employed so far ($\gamma = ^\circ\text{C} / \text{m}$).

Now γ is calculated by the following equation.

$$\gamma = \Delta t / H \text{ } ^\circ\text{C}/\text{m}$$

where Δt is the temperature difference between the lowest and highest temperature of fluid line.

For calculating Rayleigh number, the average temperature of last column i.e. ($x = 0.3\text{m}$) is calculated then the temperature difference between the average temperature and cold plate is calculated. And vertical height of cold plate is considered for calculating Rayleigh number.

$$\text{Ra}_H = g\beta H^3 (T_\alpha - T_c) / \nu^2 \times \rho_2$$

where

H is the height of the cold plate

T_c is the cold plate temperature

T_α is the average air temperature of the last column

For calculating Nusselt number, vertical height of the cold plate is considered and the value of heat transfer coefficient is considered for particular interval.

$$\text{Nu}_H = \frac{h(\tau)L}{k_f}$$

For finding out the value of heat transfer coefficient “h” for each interval the average temperature of twelve thermocouples which have been recorded through data acquisition is calculated. Then temperature difference between the average temperature and cold plate is calculated and the following equation is used.

$$Q = hA (\Delta t) \quad \text{where } \Delta t = (T_{\infty a} - T_c)$$

Findings of the experimental investigation are discussed below. The rectangular enclosure has an adiabatic surrounding, therefore, it is expected that the heat transfer rate from air to the cold plate would be unaffected by the presence of surrounding from the enclosure.

Fig. 6.1 to 6.5 show the temperature distribution of the air within the enclosure at different intervals. It is found from the figures that the temperature of air at any point gradually decreases with time and air is colder closer to the cold wall and near the bottom of the enclosure.

Again from Fig. 6.1 to 6.5 it is found that fluid layers developed with interval of time which floats on top of increasingly colder layers. It is due to the finite height of the enclosure and eventually cold boundary layer hits the bottom of the enclosure. At that point the colder stream has no choice but discharge horizontally from air of the enclosure. So, the direction of discharge is horizontal because the cold boundary layer contains fluid colder than the rest of the enclosure. In this way thermal stratification is developed in the enclosure.

From Fig. 6.6 to 6.10 it is found that the effect of the enclosure stratification on the heat transfer from air to the gradually decreasing cold wall temperature with interval of time. It is found that the value of temperature gradient, decreases with the interval of time. (Fig. 6.11). It means the temperature difference between the adjacent layer decreases with the interval of time i.e. the temperature difference among the warm fluid layers floating on top and of increasingly colder layers is decreasing and γ value is decreasing with time and approaching a constant value at each section in the enclosure.

Fig. 6.12 shows the average Nusselt number increases with the interval of time, so heat transfer coefficient also increases with the interval of time. This argument holds that heat transfer rate in the enclosure increases with time until steady state condition was reached.

The experimental data of Nusselt number and Rayleigh number is plotted in Fig. 6.13. The plots are given in term of Nusselt number versus Rayleigh number for different time of interval and for constant aspect ratio. From the graph it is found out that the relation between Nusselt number and Rayleigh number are linear. Fig. 14 shows the Rayleigh number also increases with the interval of time.

Fig. 6.15 shows the variation of temperature with respect to time. It can be seen from this graph that the temperature decreases rapidly initially. The rate of variation then decreases and finally the temperature stabilized at a steady value.

Fig. 6.16 shows air temperature difference between top most and bottom most layer of air at different x axis distance with respect to time. It can be seen from the graph that the temperature difference after some time almost same. This argument holds there is no significant change of heat transfer coefficient is established.

Fig. 6.17 shows average temperature difference of top most and bottom most layer versus time. It can be seen that after some time there is no significant change of average temperature difference with time.

Fig. 6.18 is found that the value of γ is decreasing with time. It shows that at any distance from the cold plate temperature of the stratified layers is approaching a steady state value.

CHAPTER-SEVEN

CONCLUSIONS AND RECOMMENDATIONS

7.1 Conclusions:

The important conclusions as a consequence of the present investigation are enumerated below:

1. The temperature of air at different parts of the enclosure decreases asymptotically to the steady state value.
2. Temperature difference between top most and bottom most layer of air at different x value increases initially and then remains constant.
3. The result shows that the air in a part of the enclosure is substantially stratified.
4. Stratification indicator γ decreases with the increase of time.
5. The natural convection heat transfer coefficient, Nusselt number and Rayleigh number increase with the increase of time.
6. The experimental data are well correlated with the following correlation

$$Nu_H = 0.5 \times Ra_H^{0.25}$$

7.2 Recommendations

The following recommendations are put forward as future extension of the present investigations:

1. Further investigation can be carried out with fluid filled rectangular enclosure driven by a single vertical wall with warm and cold regions:
2. A numerical model may be developed for the study of natural convection heat transfer in a rectangular enclosure from one cooled side wall.
3. For comprehensive investigation of natural convection heat transfer of the similar type, a test rig consisting of a pressure as well as a vacuum vessel may be helpful. For finding out the effect of Prandtl number experiments may be carried out with different fluids.

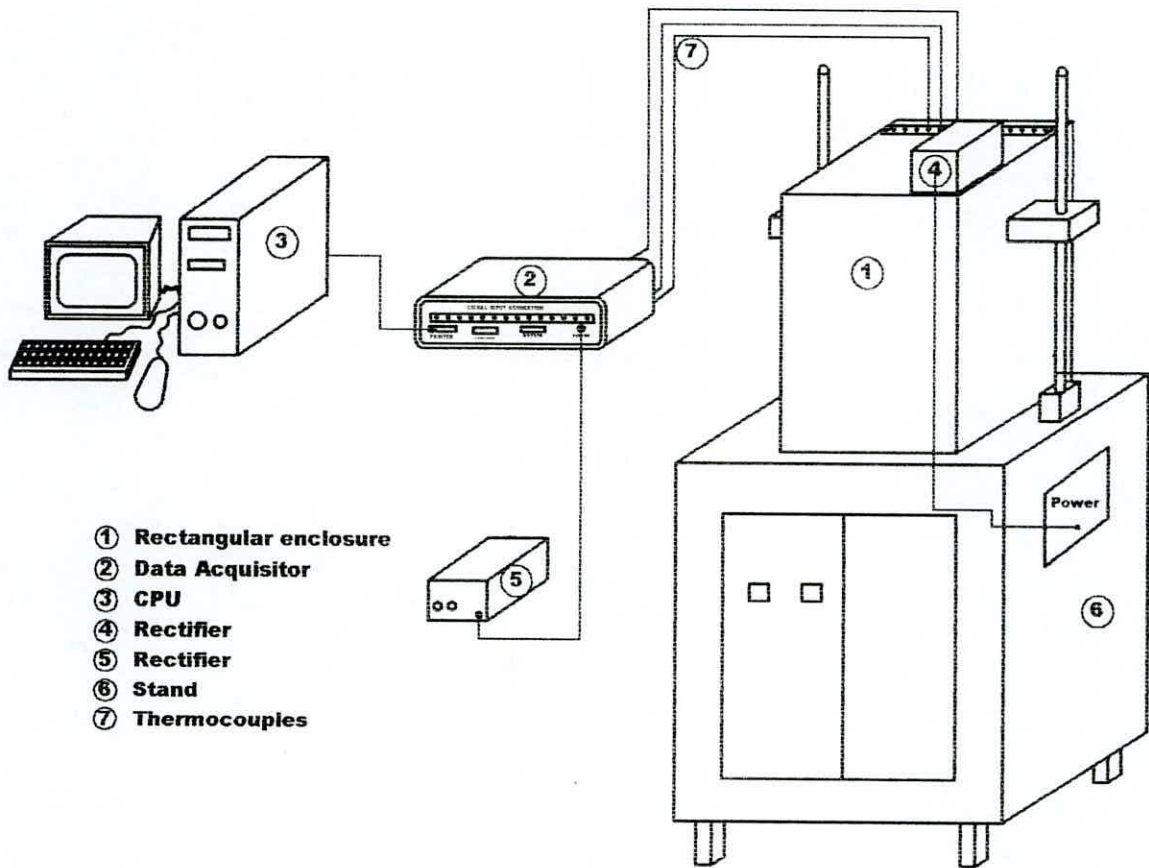
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- ① Rectangular enclosure
- ② Data Acquisitor
- ③ CPU
- ④ Rectifier
- ⑤ Rectifier
- ⑥ Stand
- ⑦ Thermocouples

Figure 4.1 : Schematic diagram of the experimental facility

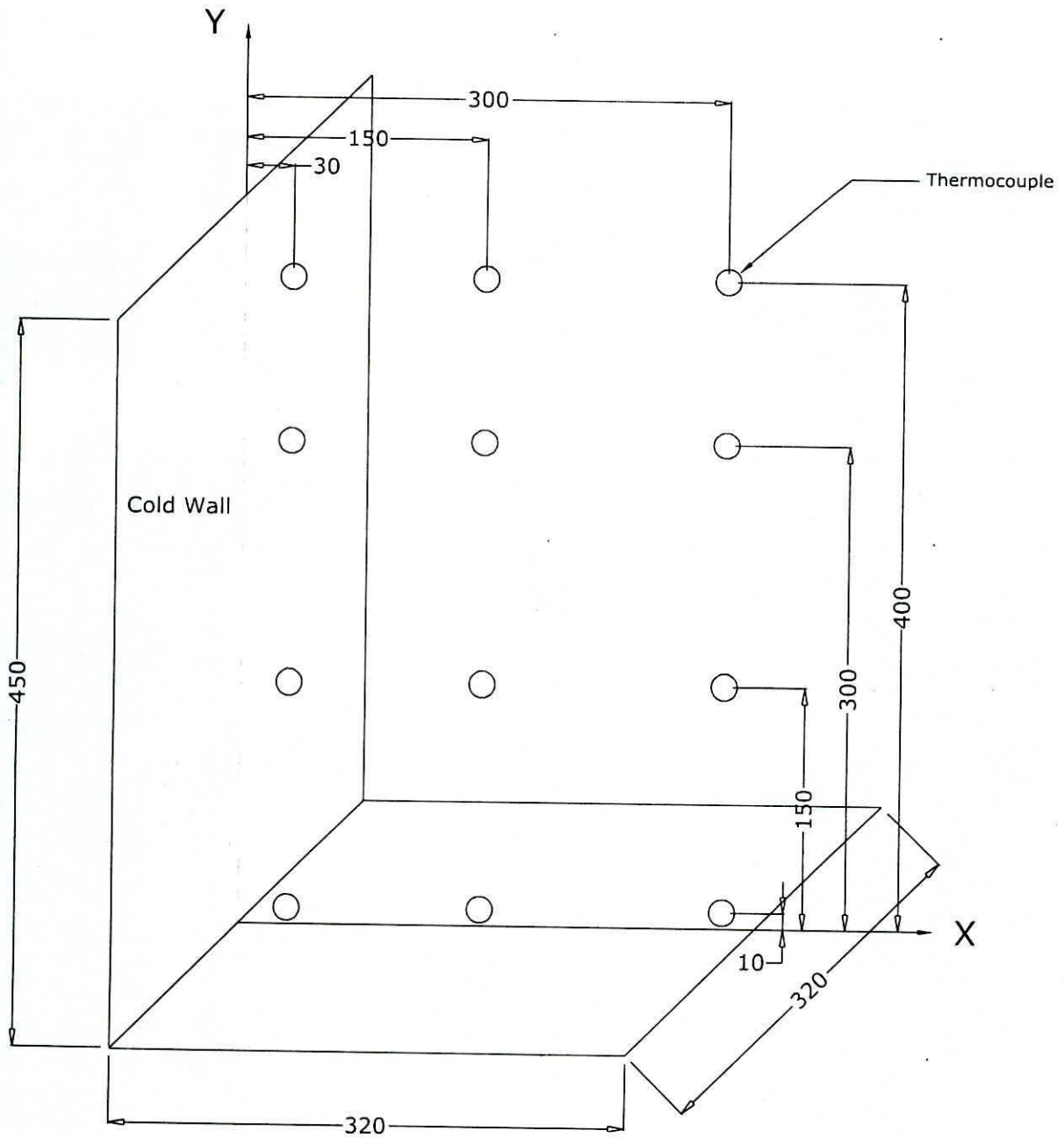


Figure 4.2: Position of the thermocouples in the enclosure

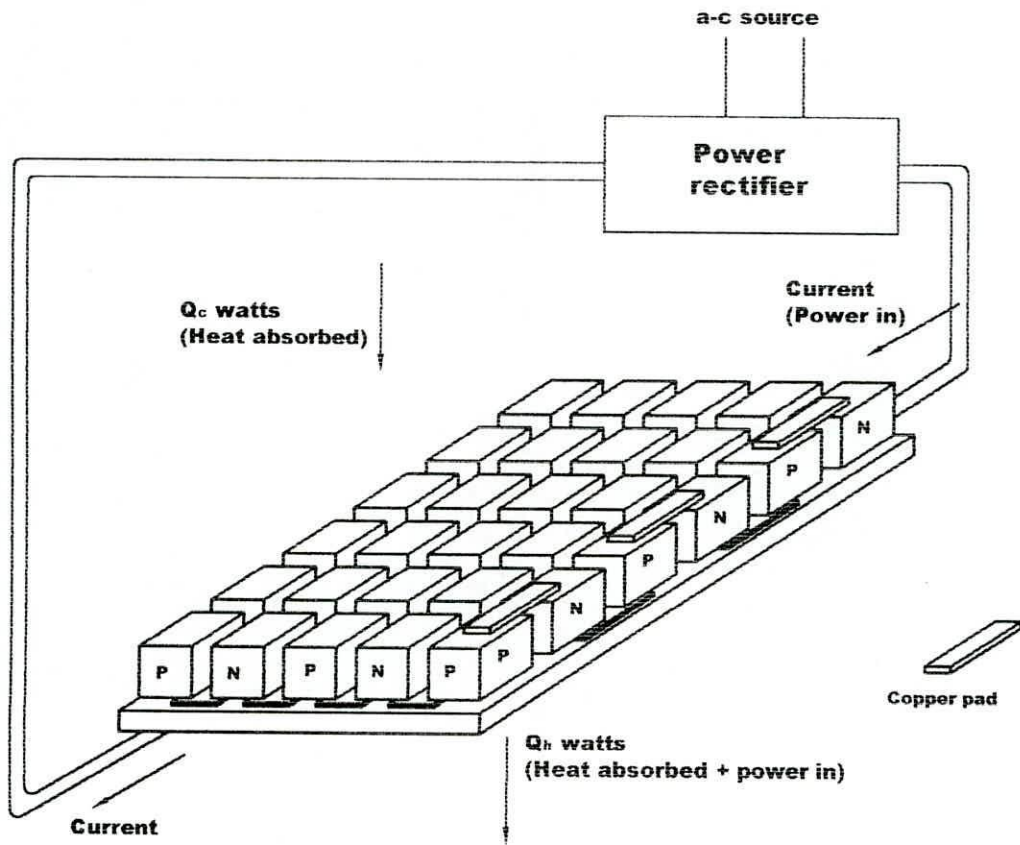
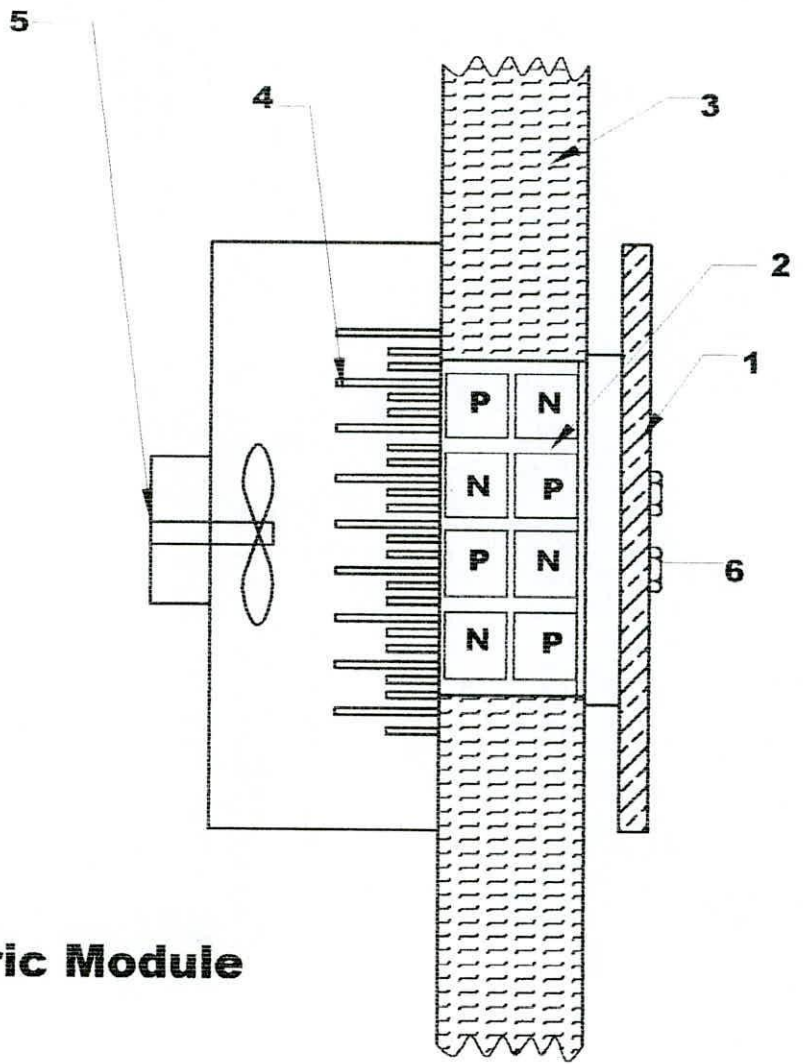
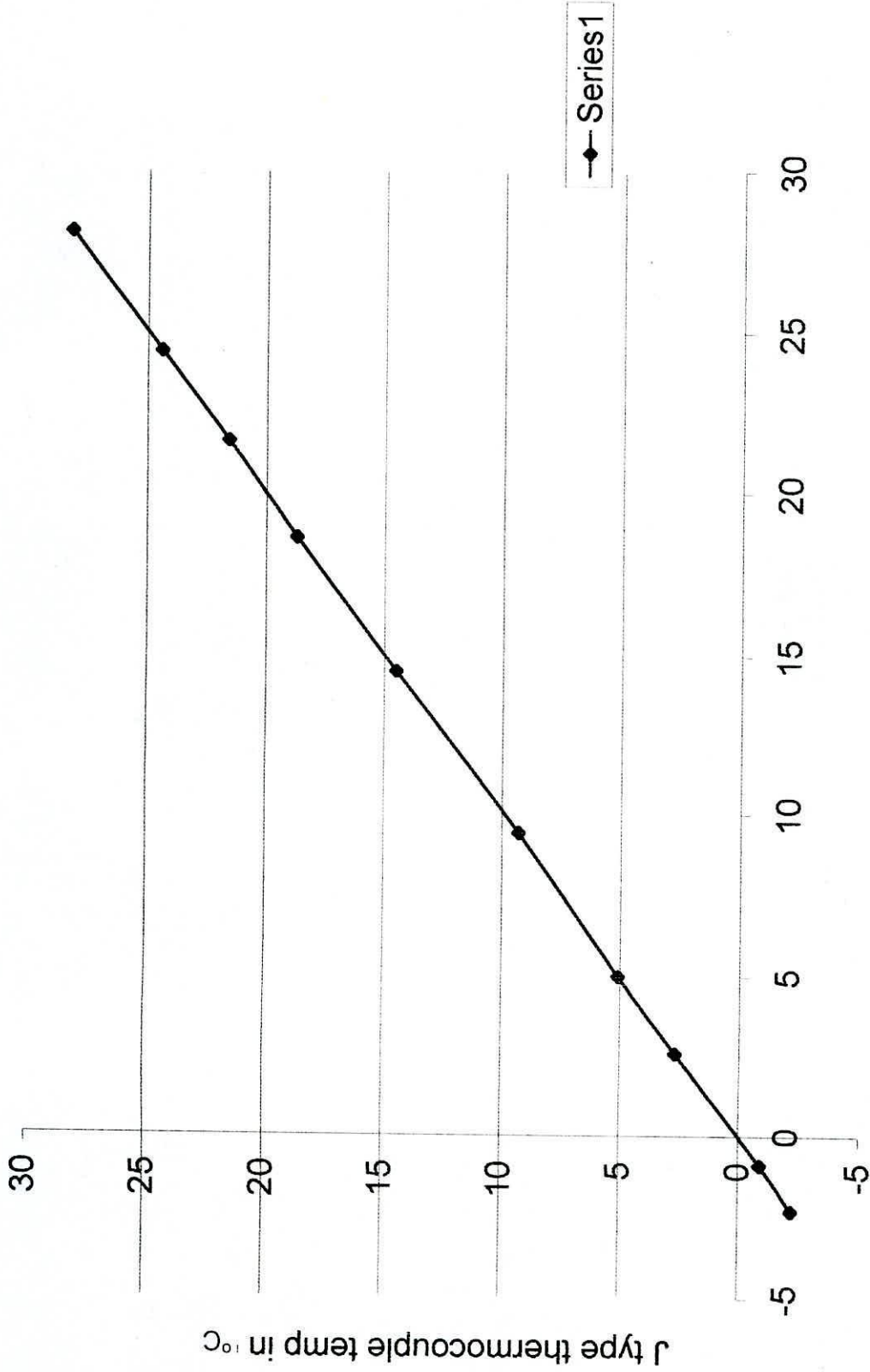


Fig. 4.3: Schematic Diagram of Thermoelectric Module with Power Supply



- ① Cold Plate
- ② Thermoelectric Module
- ③ Styrofoam
- ④ Fin
- ⑤ Fan
- ⑥ Screw

Fig. 4.4: Details of cold plate assembly



T type thermocouple temp in °C

Fig: 5.1 Calibration of thermocouple

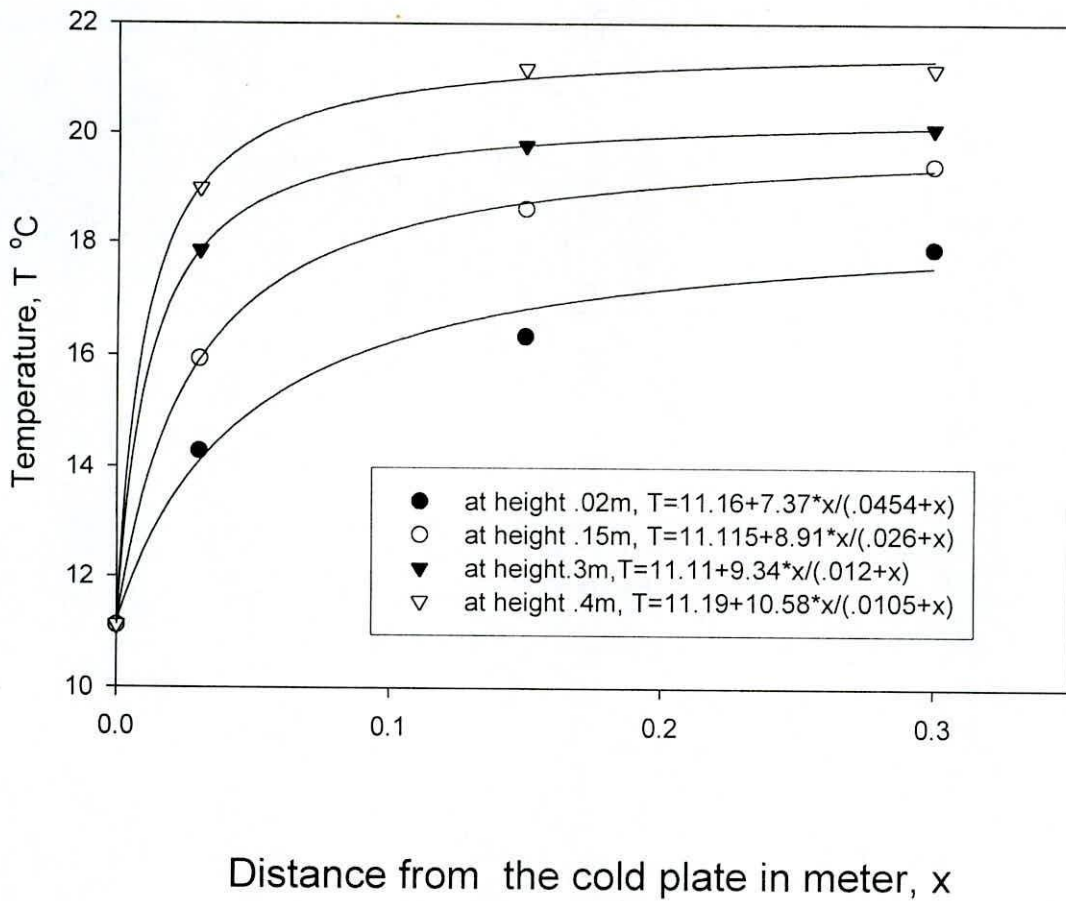


Fig. 6.1 Temperature at different height ' y ' in the enclosure at different distance ' x ' from cold plate after 15 minutes.

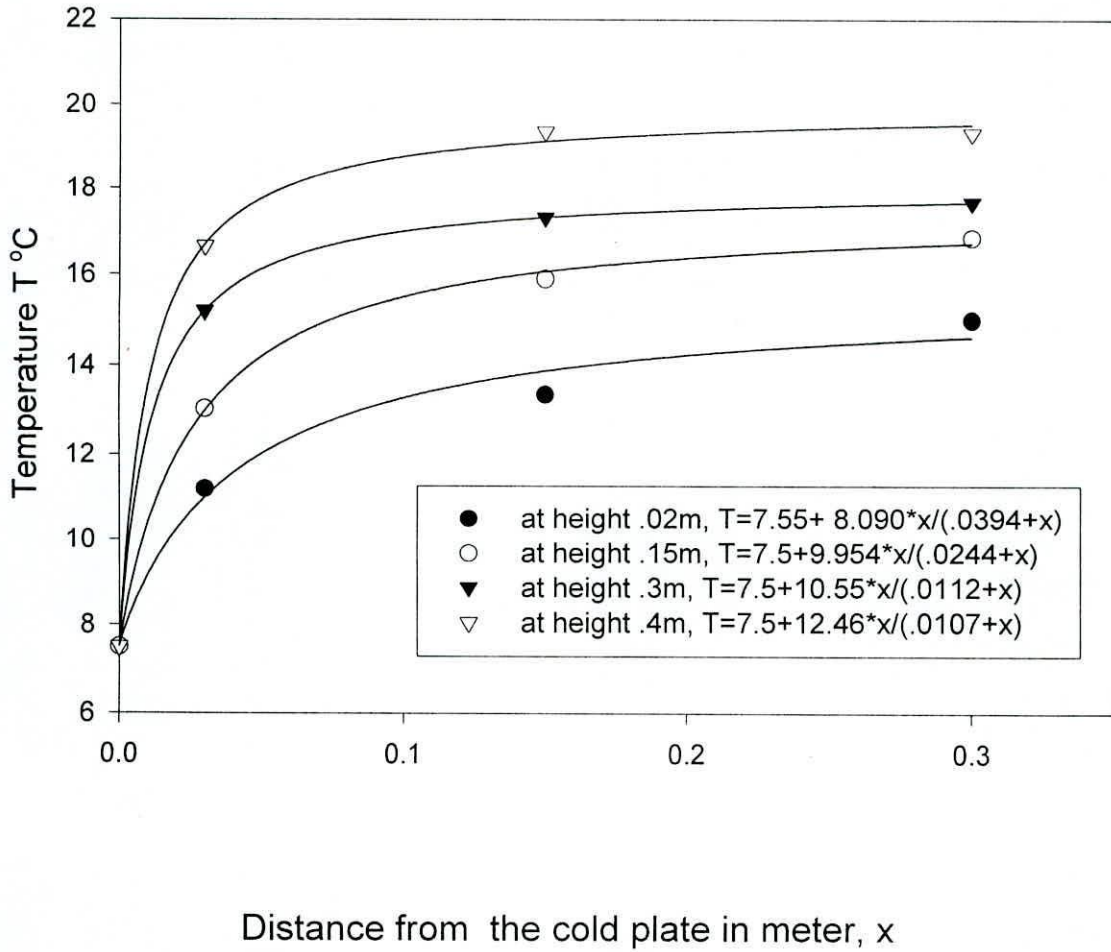


Fig. 6.2 Temperature at different height ' y ' in the enclosure at different distance ' x ' from cold plate after 25 minutes.

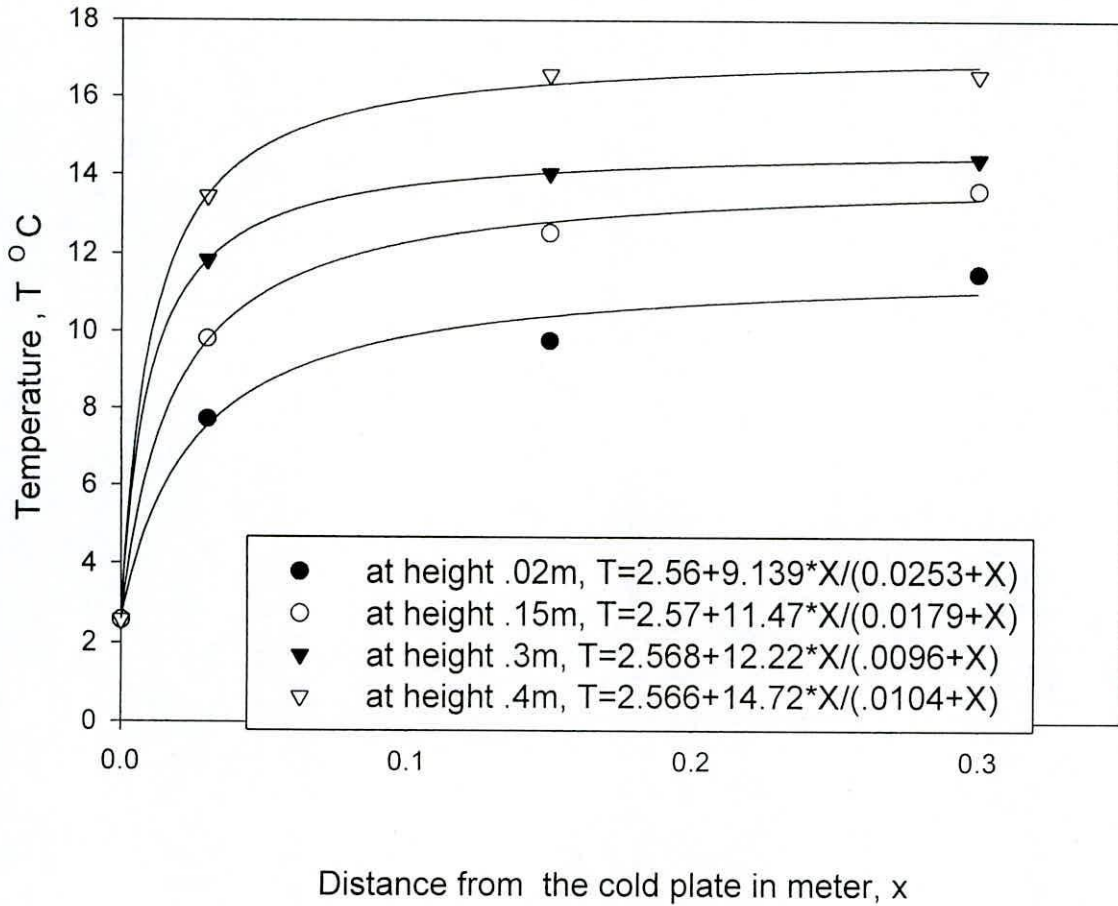


Fig. 6.3 Temperature at different height ' y ' in the enclosure at different distance ' x ' from cold plate after 50 minutes.

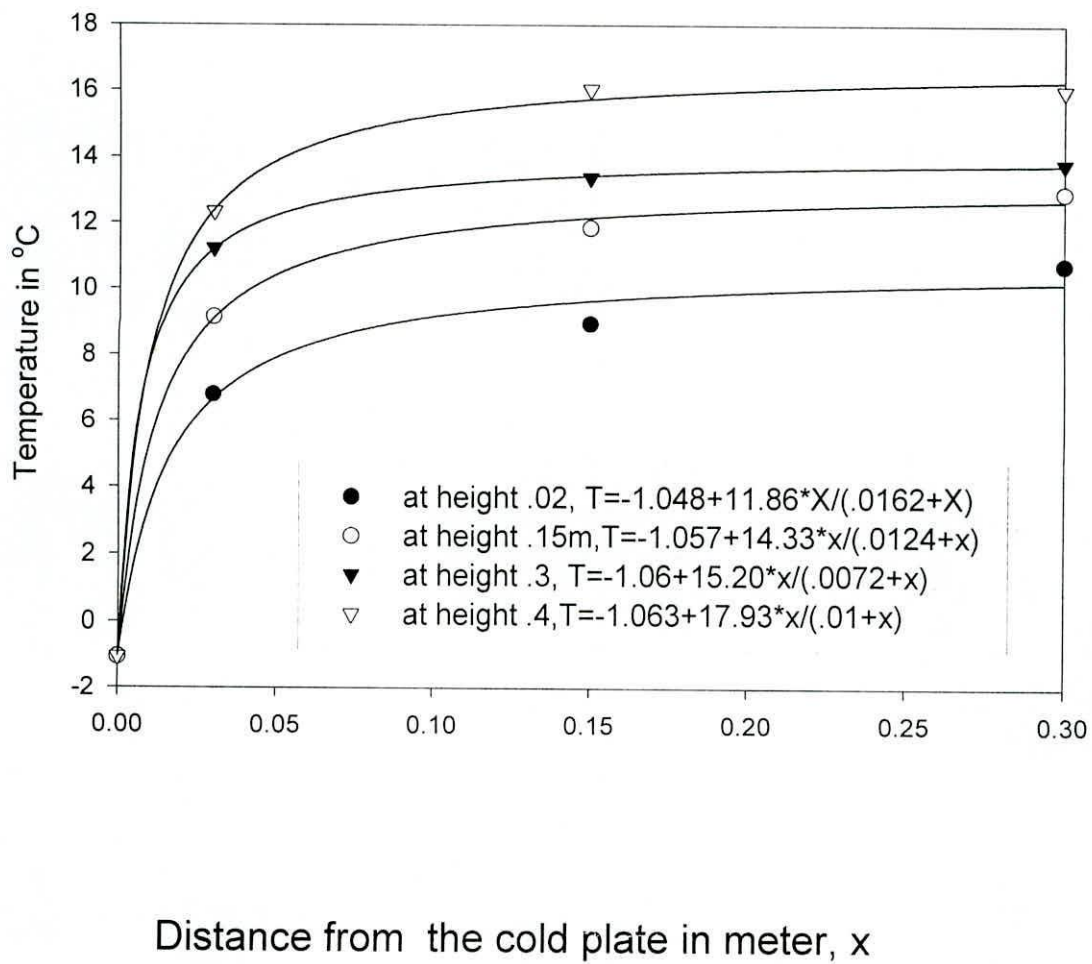


Fig. 6.4 Temperature at different height ' y ' in the enclosure at different distance ' x ' from cold plate after 60 minutes.

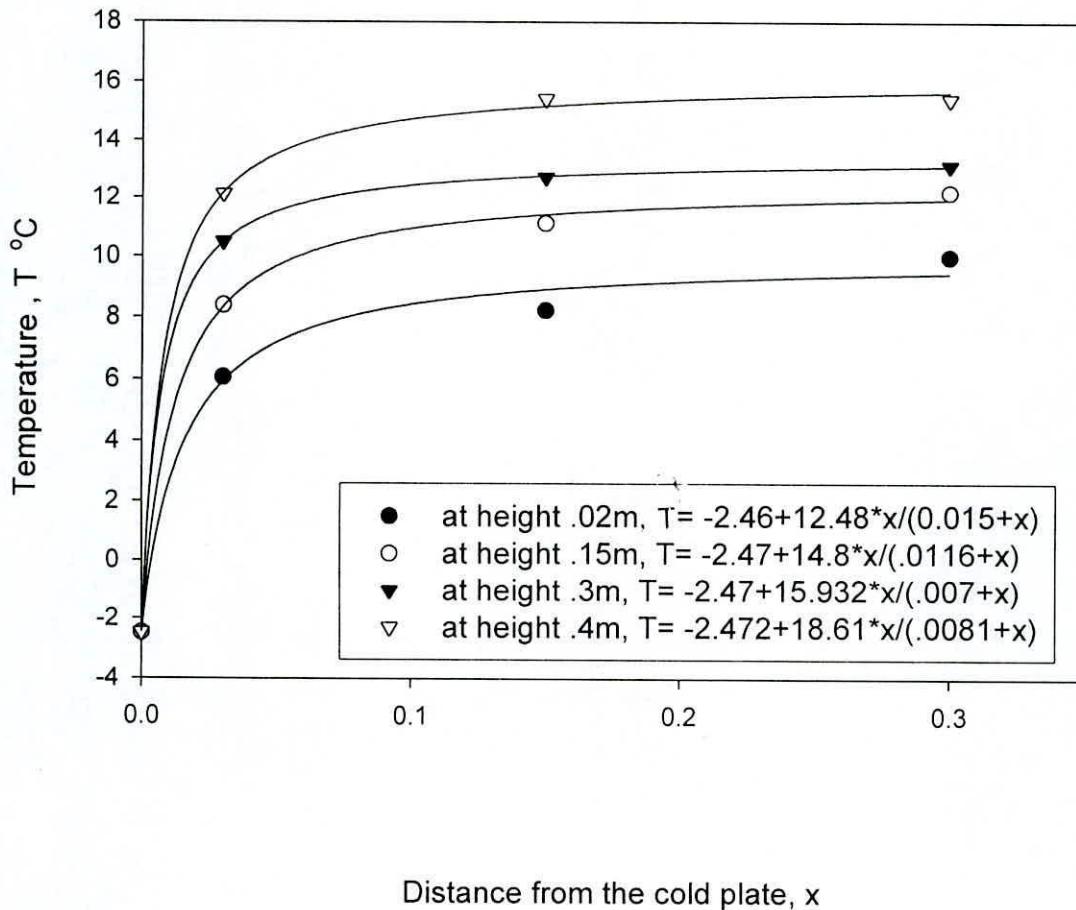


Fig.6.5 Temperature at different height ' y ' in the enclosure at different distance ' x ' from the cold plate after 80 minutes.

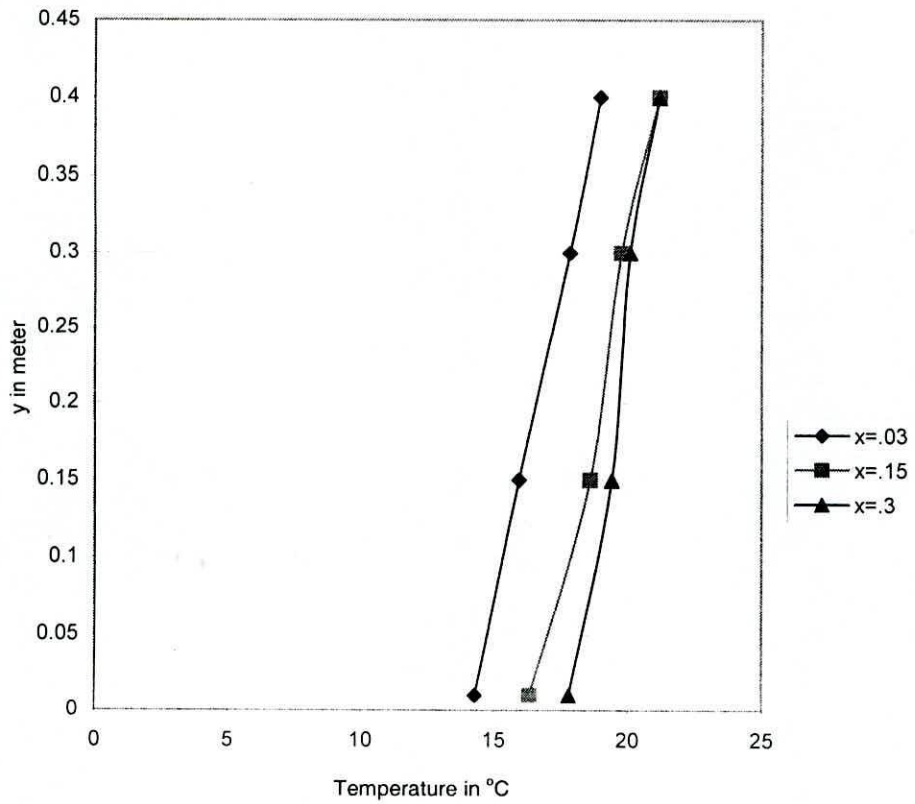


Fig. 6.6: Effect of the enclosure stratification on heat transfer from air to the cold vertical wall (after 15 mins)

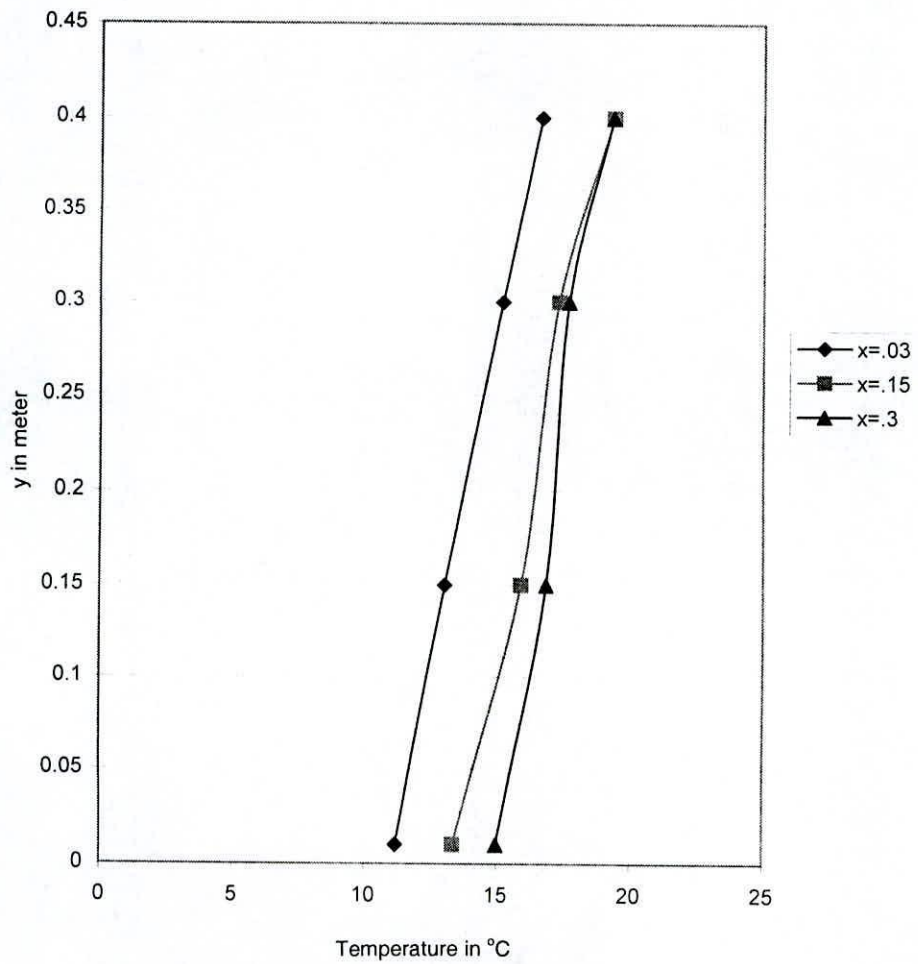


Fig. 6.7: Effect of the enclosure stratification on heat transfer from air to the cold vertical wall (after 25 mins)

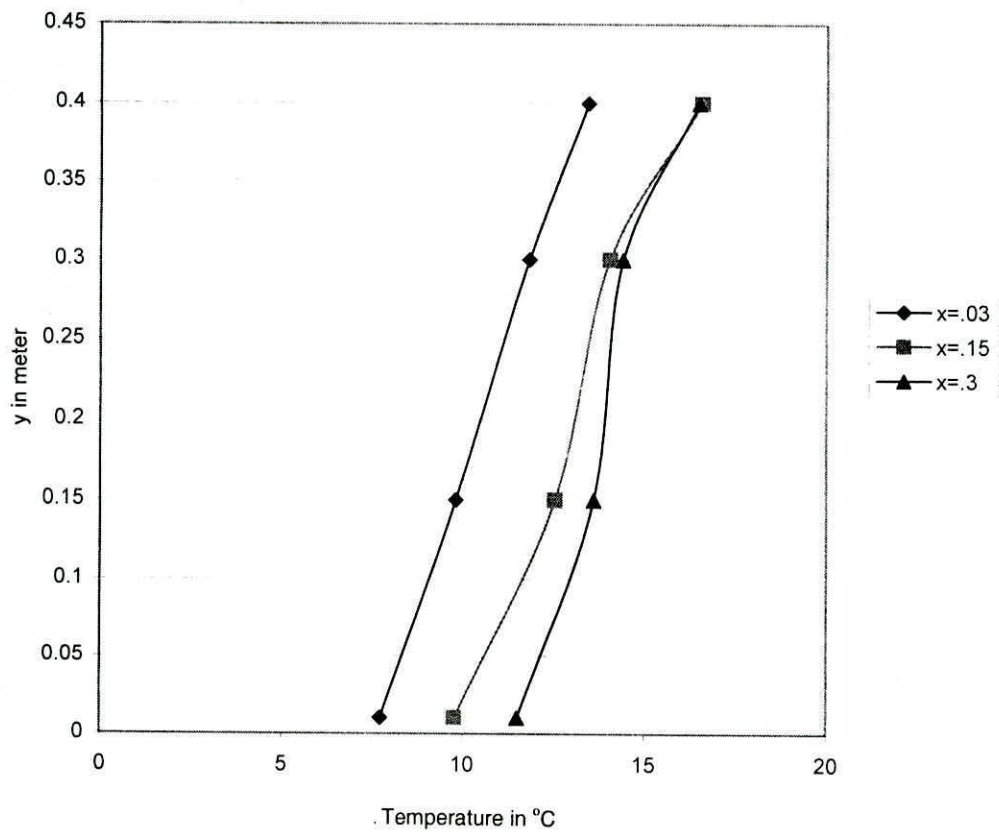


Fig. 6.8: Effect of the enclosure stratification on heat transfer from air to the cold vertical wall (after 50 mins)

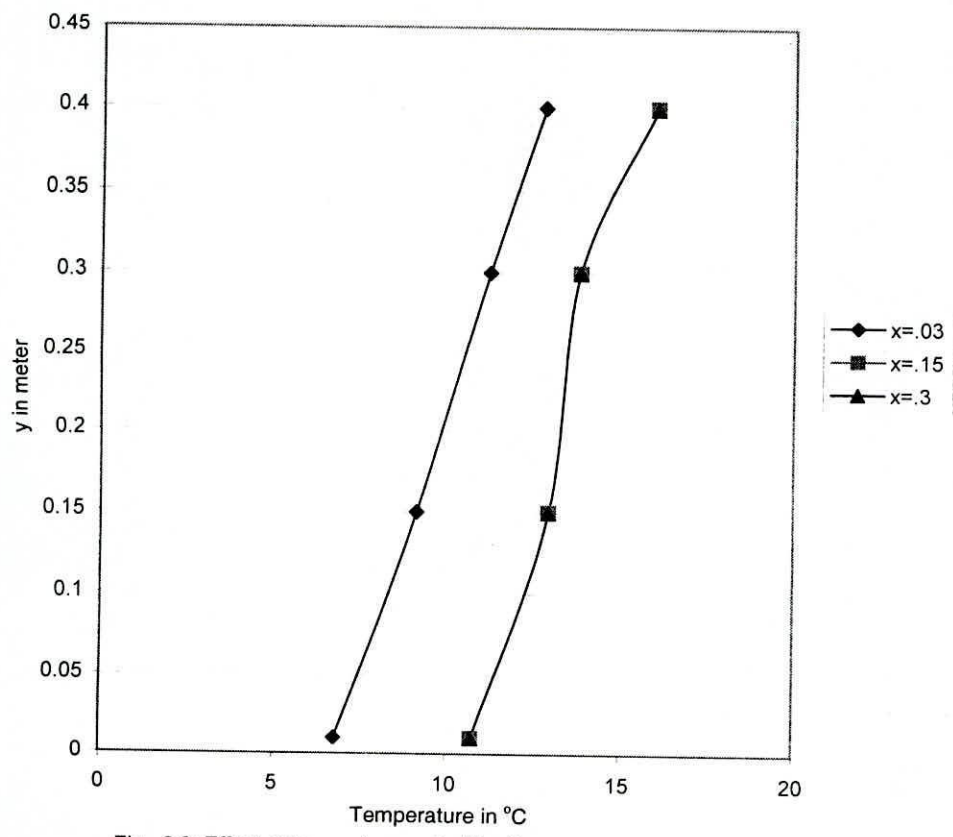


Fig. 6.9: Effect of the enclosure stratification on heat transfer from air to the cold vertical wall (after 60 mins)

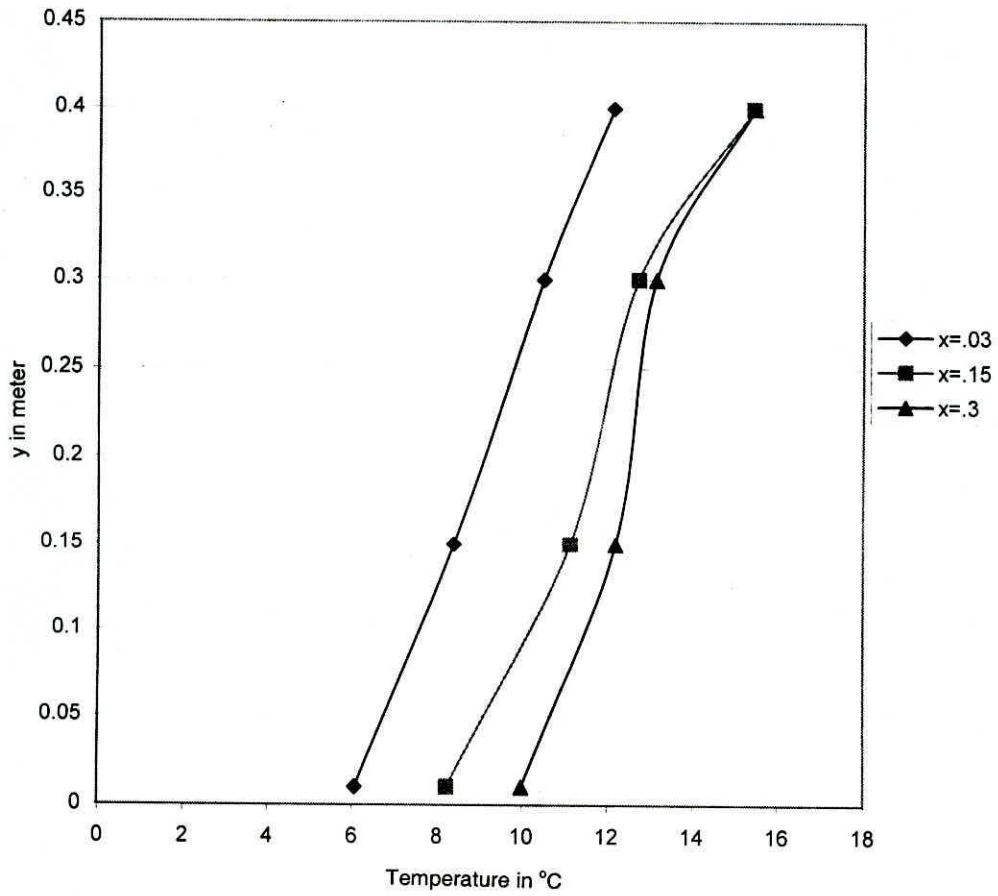


Fig. 6.10: Effect of the enclosure stratification on heat transfer from air to the cold vertical wall (after 80 mins)

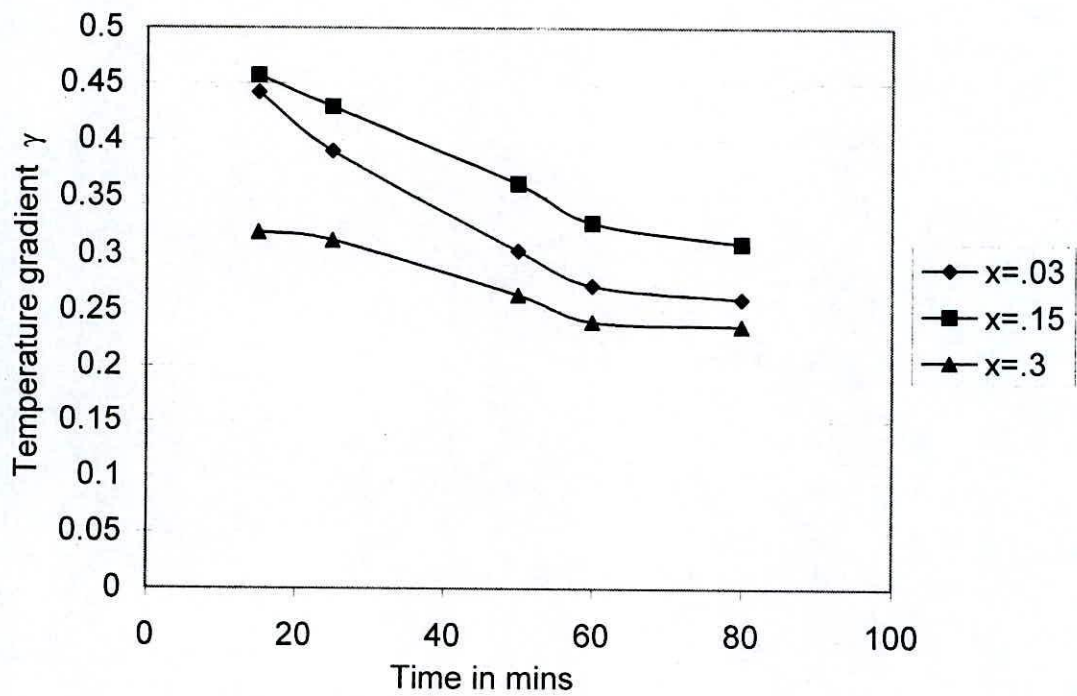


Fig 6.11 γ versus time

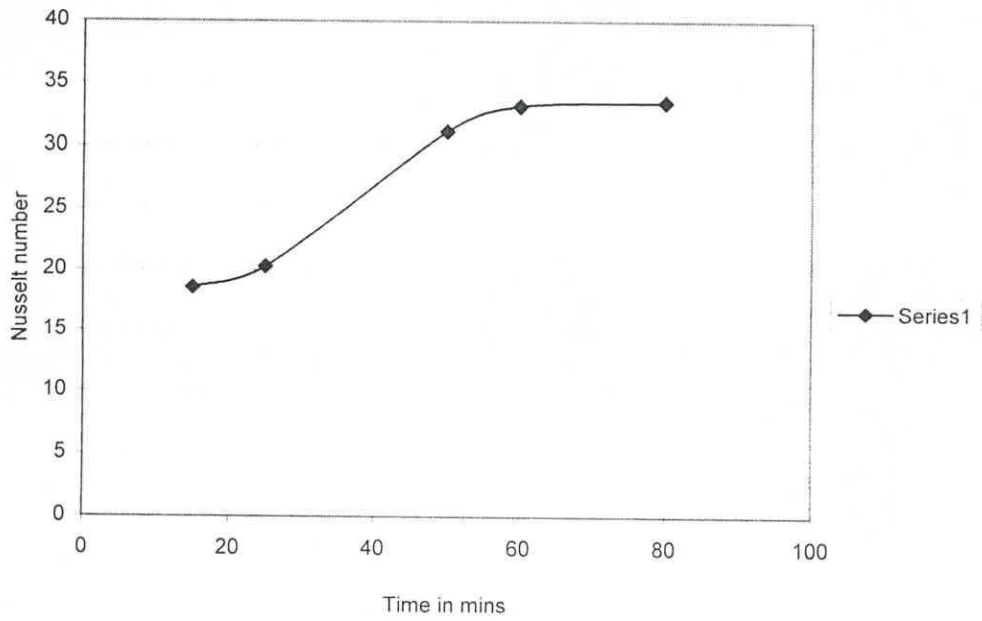


Fig: 6.12 Nusselt number versus time

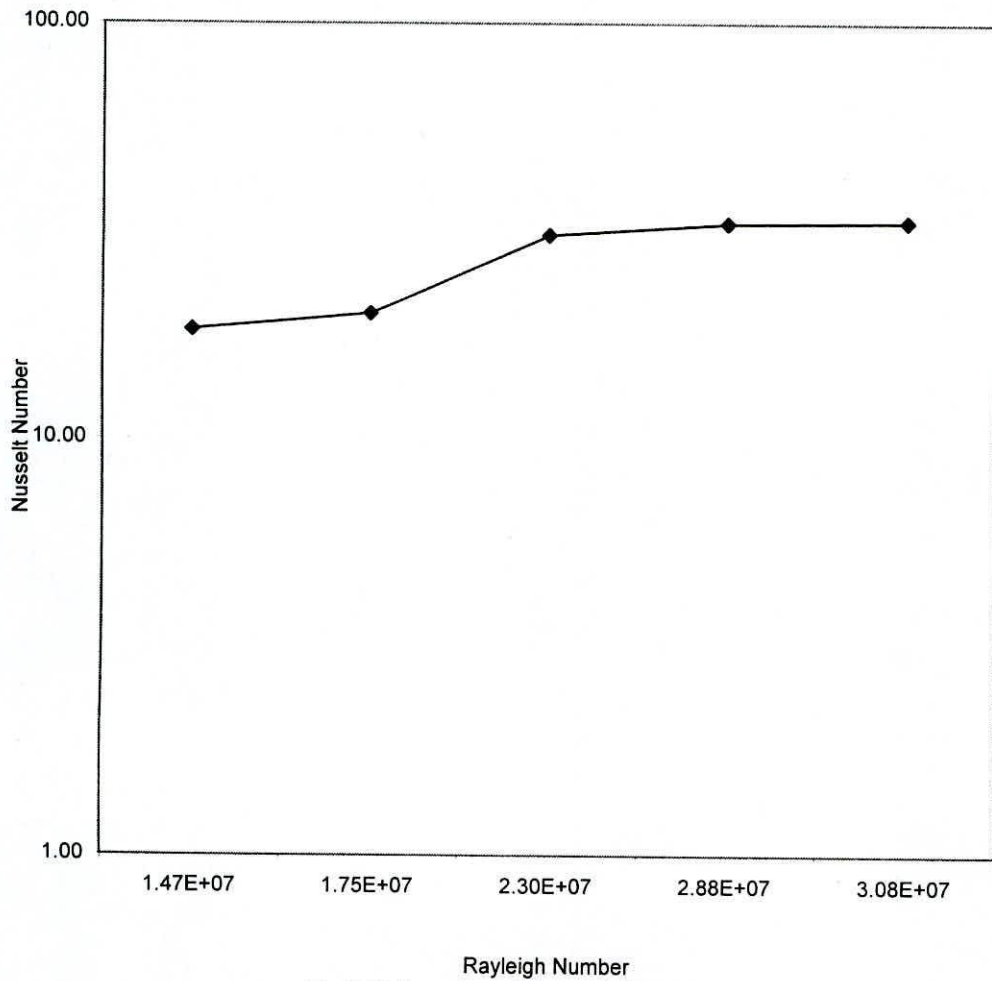


Fig. 6.13 Nusselt number versus Rayleigh number

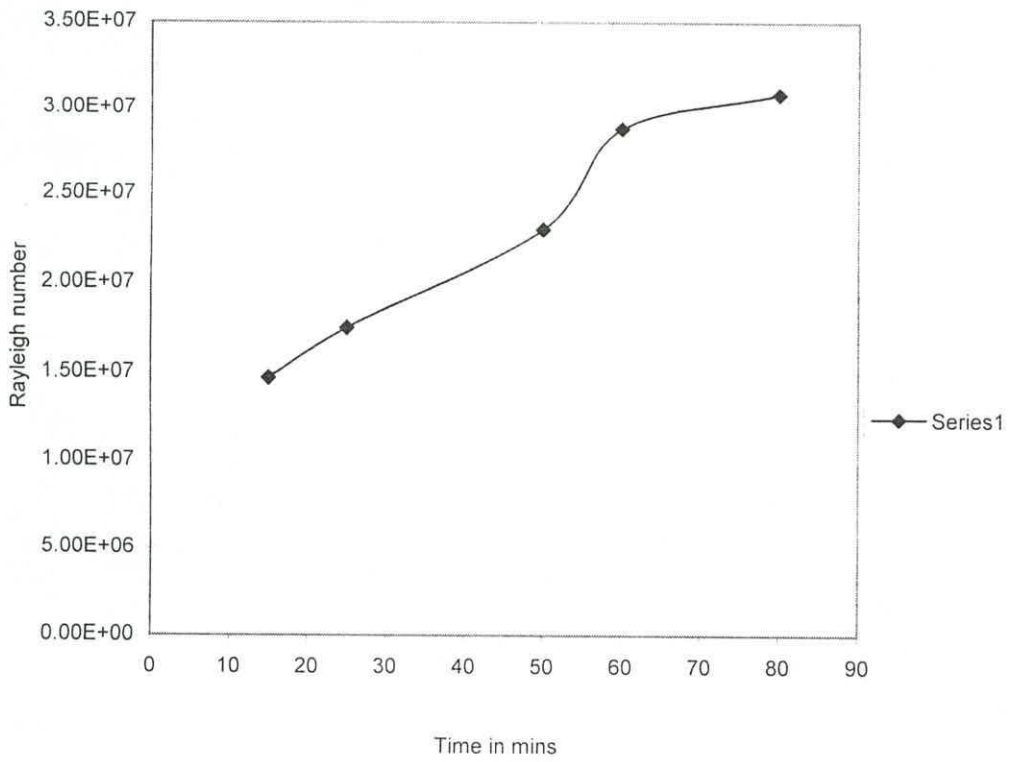


Fig:6.14 Rayleigh number versus time

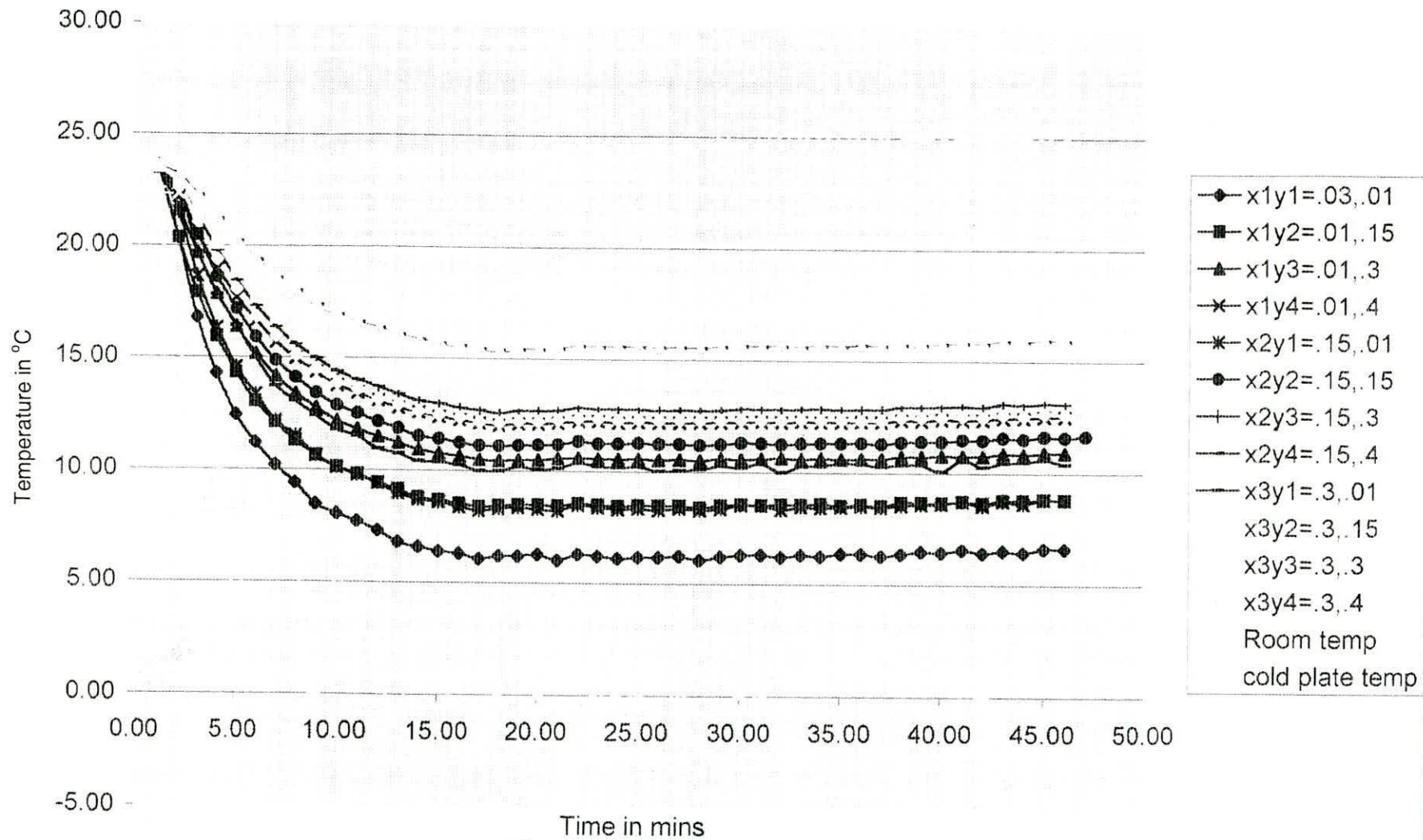


Fig. 6.15 Temperature versus time

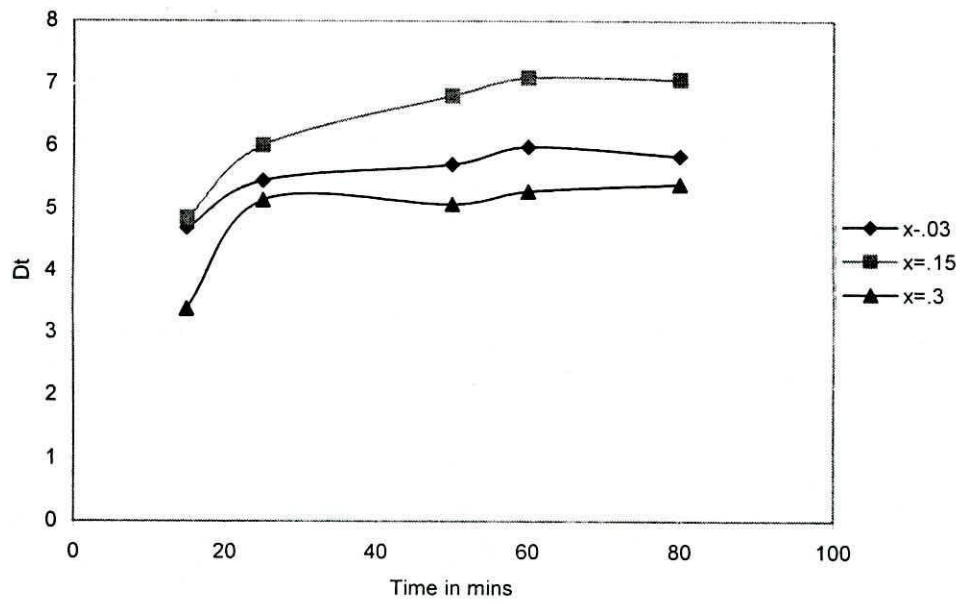


Fig.6.16 Air temperature difference between top most and bottom layer of air at different x distance with time

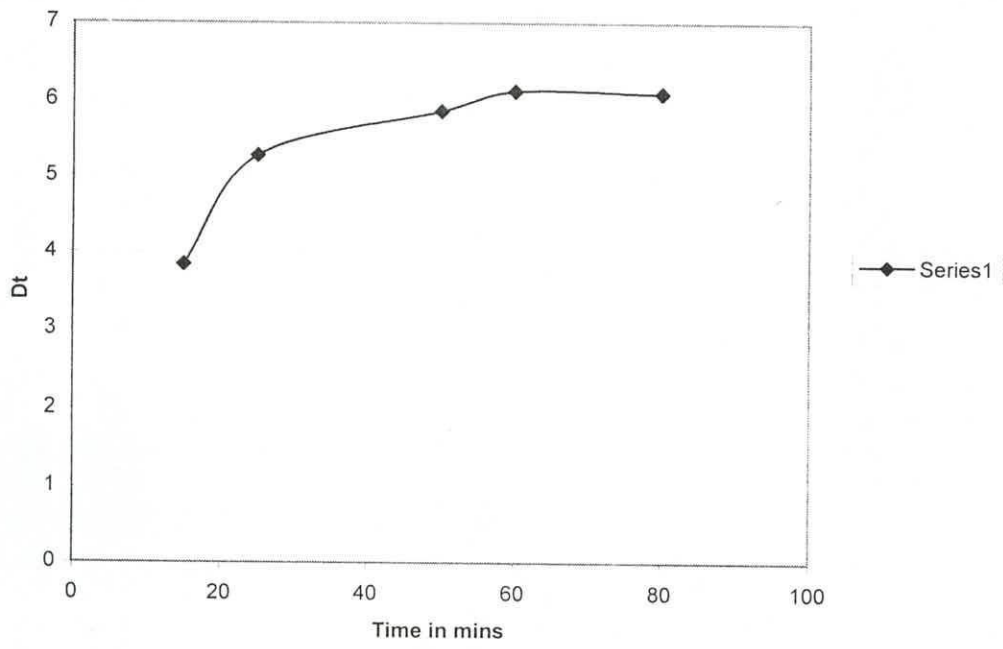


Fig:6.17 Average highest and lowest temperature difference Dt versus time

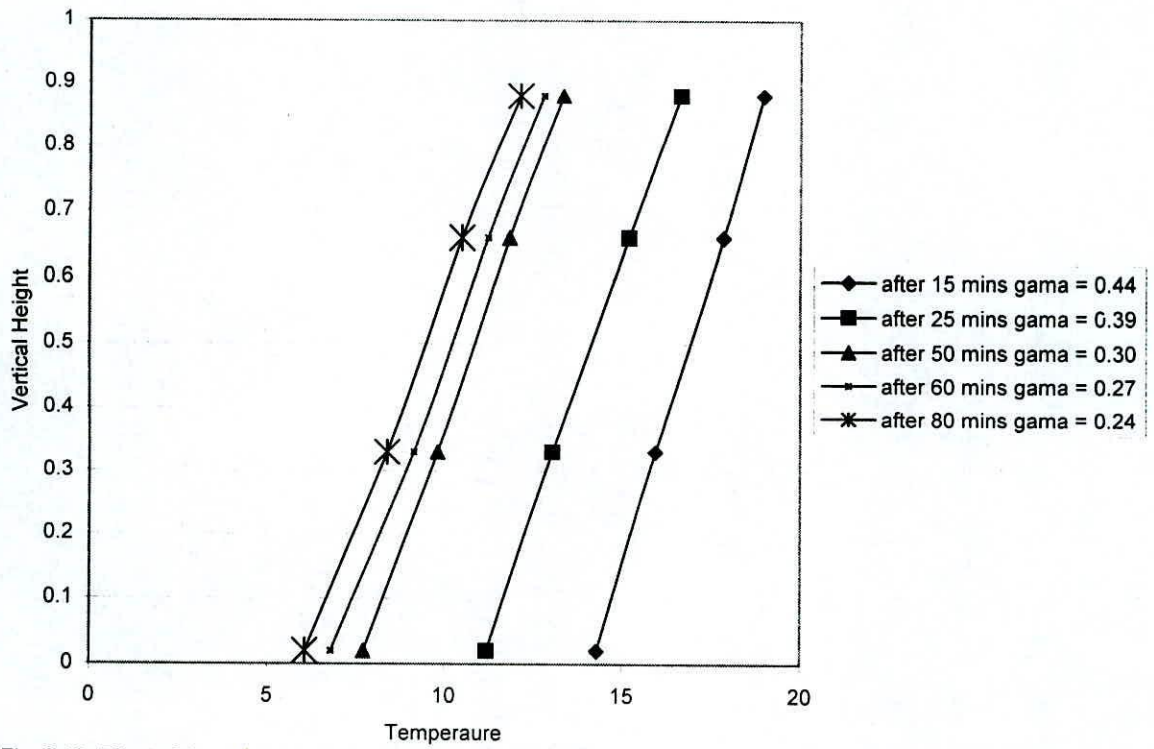


Fig. 6.18: Effect of the enclosure stratification on heat transfer from air to the cold vertical wall ($x=0.03$ m)

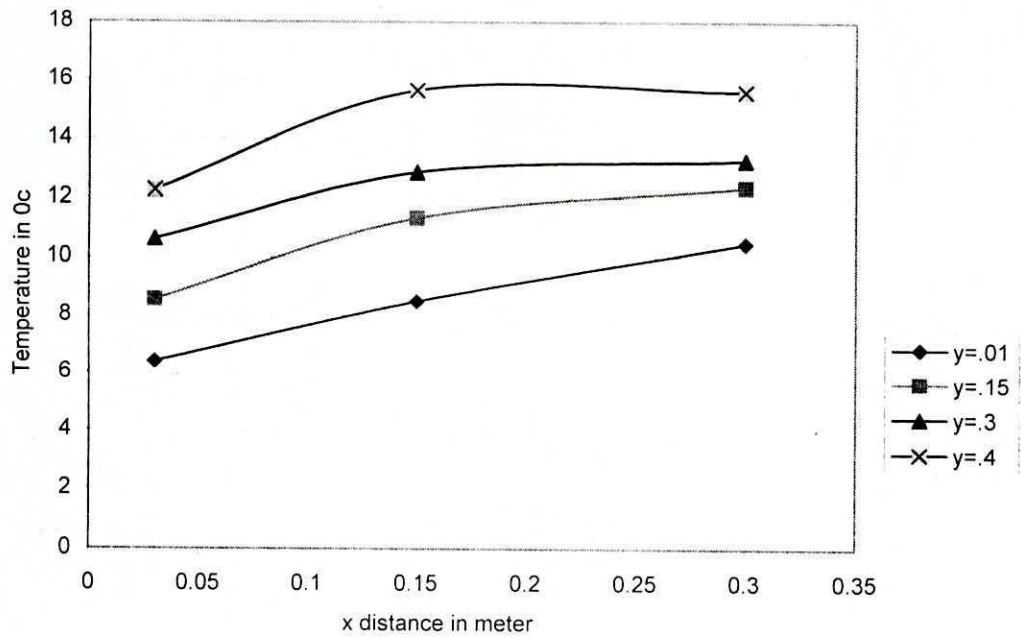


Fig:6.19 Temperature distribution of air in the enclosure at steady state condition

APPENDIX-A

Table-1 Temperature T ° C at different height from the base and at different distance from the cold plate after 15 minutes

Dist. from cold plate meter, x	Height from the base of the enclosure y in meter / Temperature				
	0.02/T °C	0.15/T °C	0.3/T °C	0.4/T °C	
0	11.11	11.11	11.11	11.11	
0.03	14.29	15.94	17.85	18.98	
0.15	16.34	18.61	19.78	21.19	
0.3	17.89	19.41	20.1	21.19	

Table-2 Temperature T ° C at different height from the base and at different distance from the cold plate after 25 minutes

Dist. from cold plate meter, x	Height from the base of the enclosure y in meter / Temperature				
	0.02/T °C	0.15/T °C	0.3/T °C	0.4/T °C	
0	7.5	7.5	7.5	7.5	
0.03	11.2	13.04	15.19	16.65	
0.15	13.36	15.9	17.31	19.37	
0.3	14.99	16.84	17.68	19.34	

Table-3 Temperature T ° C at different height from the base
and at different distance from the cold plate after 50 minutes

Dist. from cold plate meter, x	Height from the base of the enclosure y in meter / Temperature				
	0.02/T °C	0.15/T °C	0.3/T °C	0.4/T °C	
0.00	2.57	2.57	2.57	2.57	
0.03	7.73	9.81	11.83	13.44	
0.15	9.78	12.54	14.04	16.58	
0.30	11.51	13.61	14.41	16.58	

Table-4 Temperature T ° C at different height from the base
and at different distance from the cold plate after 60 minutes

Dist. from cold plate meter, x	Height from the base of the enclosure y in meter / Temperature				
	0.02/T °C	0.15/T °C	0.3/T °C	0.4/T °C	
0	-1.06	-1.06	-1.06	-1.06	
0.03	6.8	9.14	11.22	12.34	
0.15	8.96	11.88	13.4	16.05	
0.3	10.76	12.95	13.83	16.04	

Table-5 Temperature T ° C at different height from the base and at different distance from the cold plate after 80 minutes

Dist. from cold plate meter, x	Height from the base of the enclosure y in meter / Temperature				
	0.02/T °C	0.15/T °C	0.3/T °C	0.4/T °C	
0	-2.471195	-2.471195	-2.471195	-2.471195	
0.03	6.067409	8.392873	10.49329	12.13401	
0.15	8.23096	11.13024	12.71549	15.43087	
0.3	9.993936	12.19525	13.14354	15.43848	

Table-6 Critical angle for inclined rectangular cavities

(H/L)	1	3	6	12	>12
t*	25	53	60	67	70

APPENDIX-B

02-24-2003

####

14 CHAN 0 MINS

CHANNEL	1	2	3	4	5	6	7	8	9	10	11	12	13	14
Time	adj b	12 cm	24 cm	36 cm	mid b	12 cm	24 cm	36 cm	door b	12 cm	24 cm	36 cm	room	plate
HH:MM:SS	deg c	deg c	deg c	deg c	deg c	deg c	deg c	deg c	deg c	deg c	deg c	deg c	deg c	deg c
07:47:42	23.52	23.50	23.50	23.51	23.55	23.57	23.60	23.52	23.55	23.55	23.57	23.54	22.37	21.38
07:52:42	20.37	20.37	21.65	22.19	21.50	22.43	22.90	23.13	22.61	22.73	22.92	23.09	22.35	15.72
07:57:42	16.80	17.92	19.68	20.53	18.65	20.50	21.32	22.23	19.73	21.08	21.54	22.21	22.92	13.11
08:02:42	14.29	15.94	17.85	18.98	16.34	18.61	19.78	21.19	17.81	19.41	20.10	21.19	23.15	11.11
08:07:42	12.44	14.32	16.40	17.70	14.58	17.18	18.46	20.22	16.43	18.06	18.82	20.22	23.21	4.24
08:12:42	11.20	13.04	15.19	16.65	13.36	15.90	17.31	19.37	14.99	16.84	17.68	19.34	23.35	7.51
08:17:42	10.21	12.11	14.18	15.70	12.17	14.87	16.38	18.54	13.74	15.96	16.75	18.55	23.47	5.83
08:22:42	9.42	11.31	13.41	14.97	11.51	14.13	15.61	17.97	13.05	15.12	16.00	17.96	23.52	6.59
08:27:42	8.48	10.65	12.78	14.37	10.64	13.42	14.98	17.36	12.42	14.51	15.38	17.37	23.64	-0.21
08:32:42	8.06	10.13	12.18	13.78	10.11	12.88	14.41	16.94	11.79	13.88	14.80	16.93	23.84	5.34
08:37:42	7.73	9.81	11.83	13.44	9.78	12.54	14.04	16.58	11.51	13.61	14.41	16.58	23.81	2.57
08:42:42	7.32	9.44	11.50	13.11	9.43	12.18	13.73	16.33	10.95	13.18	14.11	16.30	23.98	1.67
08:47:42	6.80	9.14	11.22	12.79	8.96	11.88	13.40	16.05	10.76	12.95	13.83	16.04	23.88	-1.06
08:52:42	6.58	8.80	10.94	12.60	8.71	11.53	13.11	15.83	10.43	12.66	13.55	15.80	23.79	2.72
08:57:42	6.39	8.69	10.76	12.40	8.59	11.43	13.00	15.66	10.36	12.51	13.42	15.67	23.85	2.81
09:02:42	6.29	8.52	10.64	12.26	8.51	11.24	12.80	15.51	10.24	12.36	13.29	15.50	23.80	-2.21
09:07:42	6.07	8.39	10.49	12.13	8.23	11.13	12.72	15.43	9.99	12.20	13.14	15.44	24.10	-2.47
09:12:42	6.20	8.45	10.46	12.04	8.31	11.12	12.57	15.37	10.00	12.15	13.06	15.39	24.10	-0.11
09:17:42	6.20	8.50	10.50	12.12	8.38	11.15	12.69	15.41	10.19	12.23	13.10	15.40	24.25	1.98
09:22:42	6.27	8.49	10.50	12.12	8.30	11.17	12.70	15.41	10.07	12.23	13.16	15.41	24.32	0.07
09:27:42	6.03	8.45	10.57	12.16	8.24	11.19	12.71	15.44	10.14	12.29	13.18	15.45	24.41	0.64
09:32:42	6.29	8.57	10.59	12.25	8.49	11.35	12.84	15.58	10.39	12.36	13.25	15.57	24.56	1.33
09:37:42	6.21	8.51	10.58	12.24	8.40	11.24	12.79	15.54	10.22	12.37	13.20	15.52	24.50	-0.28
09:42:42	6.12	8.46	10.55	12.19	8.33	11.24	12.78	15.56	10.19	12.30	13.20	15.53	24.44	-0.12
09:47:42	6.19	8.50	10.57	12.21	8.36	11.28	12.79	15.54	10.22	12.32	13.18	15.52	24.39	-0.29
09:52:42	6.18	8.50	10.53	12.19	8.30	11.24	12.76	15.52	10.03	12.31	13.22	15.51	24.51	1.40
09:57:42	6.24	8.48	10.55	12.19	8.38	11.24	12.77	15.54	10.08	12.32	13.20	15.51	24.61	-0.30
10:02:42	6.06	8.41	10.51	12.21	8.30	11.21	12.74	15.53	10.13	12.32	13.20	15.50	24.69	1.20
10:07:42	6.22	8.50	10.62	12.22	8.36	11.24	12.76	15.58	10.29	12.34	13.21	15.57	24.75	1.52
10:12:42	6.29	8.56	10.58	12.25	8.53	11.33	12.85	15.61	10.21	12.36	13.26	15.63	24.67	0.15
10:17:42	6.32	8.52	10.61	12.26	8.53	11.30	12.82	15.62	10.39	12.35	13.26	15.59	24.72	-0.24
10:22:42	6.21	8.56	10.64	12.25	8.33	11.27	12.82	15.62	10.01	12.35	13.22	15.63	24.77	2.08
10:27:42	6.31	8.57	10.62	12.28	8.47	11.31	12.84	15.61	10.29	12.38	13.26	15.59	24.81	0.38
10:32:42	6.26	8.60	10.65	12.27	8.44	11.32	12.87	15.65	10.30	12.36	13.31	15.66	24.78	1.17
10:37:42	6.37	8.54	10.62	12.28	8.47	11.33	12.83	15.63	10.47	12.37	13.28	15.60	24.77	2.02
10:42:42	6.38	8.59	10.64	12.29	8.51	11.34	12.84	15.70	10.41	12.35	13.33	15.64	24.77	1.70
10:47:42	6.28	8.53	10.62	12.29	8.47	11.30	12.85	15.65	10.31	12.36	13.30	15.67	24.86	2.00
10:52:42	6.37	8.68	10.74	12.37	8.55	11.39	12.96	15.71	10.37	12.50	13.34	15.71	24.97	1.85
10:57:42	6.48	8.68	10.75	12.34	8.61	11.41	12.96	15.73	10.38	12.50	13.39	15.75	25.02	0.89
11:02:42	6.45	8.67	10.79	12.40	8.62	11.42	12.98	15.75	10.09	12.51	13.40	15.71	24.95	0.91
11:07:42	6.57	8.73	10.76	12.39	8.75	11.50	13.00	15.80	10.62	12.55	13.44	15.80	25.00	0.36
11:12:42	6.43	8.67	10.76	12.37	8.54	11.47	12.96	15.77	10.27	12.53	13.42	15.76	25.01	-0.01
11:17:42	6.52	8.79	10.88	12.48	8.76	11.60	13.11	15.89	10.48	12.64	13.54	15.86	25.06	0.34
11:22:42	6.45	8.79	10.87	12.49	8.65	11.53	13.07	15.86	10.52	12.58	13.47	15.85	25.00	1.34
11:27:42	6.62	8.85	10.91	12.55	8.84	11.65	13.13	15.94	10.61	12.70	13.57	15.91	25.25	0.38
11:32:42	6.63	8.83	10.92	12.53	8.82	11.61	13.12	15.90	10.42	12.63	13.54	15.90	25.05	2.12

11:37:42	6.64	8.89	10.97	12.57	8.84	11.66	13.20	15.98	10.62	12.77	13.57	15.97	25.14	1.33
11:42:42	6.59	8.81	10.93	12.57	8.78	11.62	13.16	15.95	10.59	12.69	13.58	15.95	25.17	1.71
11:47:42	6.64	8.90	10.94	12.60	8.79	11.62	13.15	15.98	10.57	12.64	13.61	15.96	25.42	2.01
11:52:42	6.72	8.93	11.01	12.67	8.85	11.70	13.22	16.03	10.60	12.78	13.70	16.05	25.26	2.35
11:57:42	6.72	8.95	11.04	12.72	8.80	11.74	13.28	16.11	10.52	12.78	13.74	16.07	25.39	2.12
12:02:42	6.85	9.04	11.11	12.77	9.02	11.81	13.31	16.11	10.82	12.86	13.75	16.15	25.36	2.48
12:07:42	6.76	9.08	11.12	12.77	8.90	11.83	13.34	16.10	10.49	12.84	13.77	16.14	25.32	1.54
12:12:42	6.86	9.08	11.16	12.78	9.02	11.83	13.37	16.14	10.76	12.88	13.82	16.16	25.43	0.48
12:17:42	6.84	8.99	11.14	12.79	9.00	11.83	13.35	16.19	10.89	12.89	13.85	16.18	25.39	1.24
12:22:42	6.77	9.09	11.20	12.87	8.98	11.89	13.40	16.20	10.76	12.93	13.84	16.18	25.43	2.05
12:27:42	6.90	9.10	11.24	12.87	9.06	11.92	13.44	16.25	10.74	12.96	13.87	16.25	25.44	0.38
12:32:42	6.89	9.06	11.21	12.81	9.12	11.88	13.40	16.20	10.77	12.93	13.84	16.20	25.43	0.37
12:37:42	6.91	9.08	11.21	12.88	9.07	11.89	13.43	16.22	10.79	12.97	13.86	16.25	25.62	1.63
12:42:42	6.74	9.08	11.25	12.90	8.95	11.92	13.50	16.32	10.87	13.02	13.90	16.29	25.51	1.90
12:47:42	6.93	9.14	11.30	12.93	9.17	11.97	13.48	16.33	10.93	13.06	13.94	16.29	25.53	2.45
12:52:42	6.94	9.19	11.30	12.92	9.12	12.01	13.54	16.41	10.90	13.02	13.97	16.35	25.53	1.51
12:57:42	6.91	9.17	11.30	12.96	9.08	11.99	13.53	16.37	10.71	13.10	14.01	16.36	25.54	1.53
13:02:42	7.05	9.19	11.30	12.97	9.26	12.01	13.57	16.37	11.20	13.09	13.96	16.37	25.68	0.45
13:07:42	7.03	9.22	11.34	13.01	9.22	12.03	13.61	16.44	11.08	13.12	14.04	16.41	25.57	1.37
13:12:42	7.15	9.26	11.41	13.02	9.23	12.14	13.67	16.51	10.96	13.19	14.09	16.48	25.72	2.63
13:17:42	7.01	9.21	11.37	13.07	9.24	12.07	13.66	16.42	10.97	13.13	14.09	16.45	25.75	1.39
13:22:42	7.11	9.32	11.42	13.08	9.22	12.11	13.69	16.52	10.88	13.25	14.13	16.54	25.73	0.64
13:27:42	7.10	9.29	11.44	13.10	9.23	12.10	13.67	16.54	10.73	13.17	14.13	16.54	25.85	2.67
13:32:42	7.16	9.39	11.47	13.09	9.32	12.19	13.72	16.60	11.21	13.33	14.14	16.58	25.77	0.78
13:37:42	7.09	9.39	11.49	13.19	9.34	12.26	13.76	16.60	11.17	13.29	14.21	16.64	25.76	0.82
13:42:42	7.23	9.40	11.50	13.16	9.43	12.20	13.75	16.57	11.28	13.31	14.20	16.63	25.67	2.51
13:47:42	7.21	9.47	11.54	13.19	9.42	12.22	13.81	16.64	11.36	13.30	14.25	16.67	25.86	1.22
13:52:42	7.26	9.38	11.54	13.20	9.44	12.28	13.79	16.67	11.16	13.31	14.24	16.63	25.78	2.58
13:57:42	7.23	9.46	11.58	13.26	9.37	12.31	13.82	16.67	10.97	13.36	14.25	16.64	25.95	1.67
14:02:42	7.31	9.55	11.65	13.29	9.43	12.35	13.91	16.71	11.22	13.37	14.32	16.71	25.87	1.19
14:07:42	7.18	9.51	11.67	13.28	9.34	12.39	13.91	16.72	11.34	13.43	14.36	16.75	25.84	2.83
14:12:42	7.30	9.58	11.64	13.31	9.40	12.35	13.91	16.72	11.24	13.46	14.32	16.72	25.98	2.74
14:17:42	7.39	9.51	11.69	13.30	9.56	12.39	13.95	16.72	11.17	13.46	14.37	16.72	25.79	1.94
14:22:42	7.53	9.65	11.73	13.35	9.63	12.45	13.98	16.78	11.25	13.52	14.38	16.77	25.99	2.10
14:27:42	7.44	9.57	11.69	13.35	9.62	12.38	13.96	16.75	11.39	13.50	14.41	16.74	25.71	0.76
14:32:42	7.46	9.71	11.71	13.39	9.66	12.45	13.98	16.82	11.57	13.58	14.37	16.81	25.77	2.04
14:37:42	7.32	9.63	11.72	13.39	9.50	12.41	14.01	16.88	11.23	13.57	14.42	16.87	25.87	2.82
14:42:42	7.55	9.67	11.76	13.47	9.66	12.47	14.02	16.89	11.28	13.48	14.45	16.89	25.96	3.08

02-26-2003	0.29														14.00 CHANN	0.00 MINS
CHANNEL	1.00	2.00	3.00	4.00	5.00	6.00	7.00	8.00	9.00	10.00	11.00	12.00	13.00	14.00		
Time	2mm	150mm	300mm	400mm	2mm	150mm	300mm	400mm	2mm	150mm	300mm	400mm	room	plate		
HH:MM:SS	deg c	deg c	deg c	deg c	deg c	deg c	deg c	deg c	deg c	deg c	deg c	deg c	deg c	deg c		
07:02:25	25.30	25.31	25.26	25.30	25.25	25.29	25.32	25.24	25.25	25.24	25.26	25.19	23.06	28.06		
07:07:25	22.13	22.14	23.28	23.79	23.15	24.10	24.56	24.71	24.27	24.38	24.57	24.71	23.52	16.27		
07:12:25	18.42	19.46	21.15	21.94	20.27	21.99	22.85	23.68	21.80	22.62	22.98	23.67	23.54	15.15		
07:17:25	15.79	17.23	19.18	20.34	18.01	19.96	21.12	22.51	19.50	20.77	21.41	22.50	24.09	12.57		
07:22:25	13.78	15.52	17.60	18.97	16.03	18.35	19.67	21.48	17.70	19.22	20.04	21.45	23.92	10.23		
07:27:25	12.28	14.25	16.30	17.73	14.54	17.09	18.44	20.51	16.06	18.05	18.86	20.51	24.08	9.43		
07:32:25	11.21	13.11	15.24	16.77	13.47	15.93	17.45	19.62	15.06	16.97	17.85	19.61	24.08	4.79		
07:37:25	10.30	12.26	14.34	15.91	12.62	15.12	16.58	18.94	14.48	16.13	16.98	18.92	24.12	7.21		
07:42:25	9.53	11.58	13.68	15.25	11.78	14.34	15.91	18.33	13.49	15.38	16.30	18.30	24.06	3.07		
07:47:25	8.81	11.00	13.11	14.71	11.05	13.80	15.31	17.83	12.80	14.82	15.76	17.81	24.27	0.80		
07:52:25	8.25	10.49	12.65	14.23	10.62	13.26	14.84	17.44	12.63	14.35	15.29	17.39	24.32	0.26		
07:57:25	8.02	10.20	12.31	13.92	10.31	12.96	14.54	17.14	12.03	13.98	14.97	17.16	24.60	-0.62		
08:02:25	7.74	9.90	12.01	13.63	9.99	12.68	14.21	16.92	11.76	13.80	14.66	16.90	24.71	5.01		
08:07:25	7.51	9.80	11.81	13.47	9.84	12.60	14.07	16.75	11.33	13.60	14.51	16.75	24.83	4.91		
08:12:25	7.45	9.64	11.66	13.30	9.70	12.37	13.86	16.63	11.32	13.40	14.37	16.65	24.99	4.29		
08:17:25	7.32	9.58	11.62	13.27	9.50	12.27	13.83	16.62	11.19	13.41	14.29	16.58	25.00	-0.96		
08:22:25	7.17	9.51	11.58	13.18	9.40	12.25	13.78	16.53	10.76	13.27	14.21	16.52	25.02	3.96		
08:27:25	7.14	9.52	11.51	13.14	9.41	12.28	13.77	16.51	11.40	13.25	14.17	16.49	24.85	5.01		
08:32:25	7.16	9.43	11.45	13.07	9.47	12.17	13.67	16.43	11.24	13.14	14.08	16.40	25.13	5.01		
08:37:25	6.95	9.37	11.39	13.04	9.27	12.14	13.62	16.41	11.12	13.18	14.07	16.41	25.29	4.11		
08:42:25	7.07	9.29	11.41	13.07	9.32	12.09	13.63	16.43	10.88	13.12	14.07	16.40	24.98	3.15		
08:47:25	6.88	9.21	11.36	13.07	9.29	12.14	13.63	16.37	11.16	13.18	14.06	16.38	25.33	-1.80		
08:52:25	6.76	9.19	11.36	13.01	9.21	12.02	13.56	16.36	10.98	13.07	14.01	16.33	25.29	-1.62		
08:57:25	6.90	9.28	11.35	12.98	9.28	12.10	13.56	16.43	10.93	13.12	14.05	16.41	25.54	4.52		
09:02:25	6.96	9.32	11.35	13.01	9.36	12.03	13.61	16.46	11.21	13.04	14.09	16.43	25.59	4.38		
09:07:25	7.02	9.35	11.47	13.07	9.48	12.18	13.71	16.44	11.32	13.23	14.17	16.46	25.68	-1.15		
09:12:25	7.08	9.36	11.51	13.15	9.39	12.20	13.72	16.54	11.03	13.26	14.16	16.54	25.46	-1.08		
09:17:25	6.97	9.47	11.57	13.15	9.41	12.27	13.80	16.56	11.29	13.37	14.23	16.53	25.64	4.74		
09:22:25	7.18	9.49	11.57	13.22	9.46	12.27	13.81	16.62	11.22	13.39	14.24	16.55	25.78	-1.11		
09:27:25	7.11	9.48	11.64	13.25	9.46	12.35	13.86	16.63	11.35	13.37	14.31	16.66	25.89	-1.15		
09:32:25	7.33	9.61	11.71	13.38	9.71	12.44	13.95	16.76	11.52	13.47	14.44	16.77	25.98	4.54		
09:37:25	7.21	9.65	11.76	13.37	9.57	12.47	13.96	16.78	11.38	13.52	14.40	16.77	26.04	2.99		
09:42:25	7.26	9.67	11.75	13.37	9.69	12.53	14.02	16.82	11.48	13.56	14.44	16.81	26.12	-0.45		
09:47:25	7.47	9.74	11.85	13.48	9.77	12.51	14.09	16.89	11.51	13.59	14.56	16.86	26.18	0.11		
09:52:25	7.49	9.83	11.96	13.56	9.83	12.60	14.18	16.97	11.49	13.67	14.61	16.96	26.29	-0.29		
09:57:25	7.50	9.81	11.92	13.60	9.86	12.61	14.18	17.00	11.68	13.66	14.65	16.96	26.33	1.97		
10:02:25	7.64	9.88	11.97	13.63	9.92	12.69	14.24	17.00	11.42	13.81	14.69	17.00	26.31	-0.70		
10:07:25	7.56	9.97	12.07	13.70	9.81	12.77	14.30	17.08	11.47	13.87	14.74	17.08	26.20	4.31		
10:12:25	7.65	10.07	12.15	13.75	10.04	12.90	14.38	17.17	11.81	13.89	14.80	17.15	26.42	2.64		
10:17:25	7.73	9.99	12.06	13.76	10.02	12.78	14.32	17.13	11.71	13.81	14.79	17.10	26.28	2.09		
10:22:25	7.61	10.01	12.11	13.80	9.89	12.78	14.34	17.18	11.65	13.87	14.84	17.16	26.36	4.86		
10:27:25	7.83	10.09	12.14	13.78	10.12	12.91	14.42	17.18	11.96	13.94	14.83	17.18	26.55	2.48		
10:32:25	7.76	10.06	12.21	13.84	10.10	12.90	14.45	17.22	11.76	13.97	14.91	17.24	26.43	1.49		
10:37:25	7.81	10.18	12.27	13.87	10.17	13.05	14.52	17.29	11.84	14.02	14.99	17.27	26.46	2.57		
10:42:25	7.77	10.11	12.24	13.93	10.09	12.94	14.52	17.31	11.69	14.03	14.99	17.30	26.46	3.56		
10:47:25	7.88	10.29	12.34	13.96	10.28	13.01	14.57	17.38	12.06	14.08	15.05	17.36	26.59	5.62		

10:52:25	7.82	10.24	12.34	14.00	10.16	13.08	14.58	17.38	12.08	14.10	15.06	17.37	26.62	3.68
10:57:25	7.92	10.29	12.39	14.03	10.28	13.13	14.64	17.44	11.93	14.21	15.07	17.44	26.68	2.75
11:02:25	8.10	10.42	12.46	14.08	10.41	13.14	14.73	17.48	12.15	14.24	15.12	17.47	26.63	5.77
11:07:25	8.06	10.37	12.50	14.08	10.38	13.22	14.71	17.49	12.15	14.26	15.14	17.49	26.75	2.82
11:12:25	8.07	10.45	12.53	14.11	10.40	13.21	14.73	17.53	12.26	14.31	15.20	17.53	26.82	3.38
11:17:25	8.09	10.50	12.57	14.22	10.46	13.25	14.78	17.58	12.26	14.36	15.27	17.54	26.64	3.76
11:22:25	8.33	10.61	12.63	14.26	10.66	13.36	14.90	17.63	12.37	14.42	15.32	17.63	26.89	4.36
11:27:25	8.37	10.65	12.70	14.35	10.67	13.45	14.94	17.71	12.37	14.45	15.39	17.70	26.88	1.98
11:32:25	8.32	10.60	12.72	14.33	10.63	13.40	14.93	17.70	12.36	14.46	15.39	17.70	26.66	3.83
11:37:25	8.39	10.60	12.74	14.31	10.66	13.43	14.94	17.72	12.32	14.51	15.41	17.71	26.89	2.99
11:42:25	8.48	10.75	12.83	14.39	10.81	13.53	15.02	17.77	12.66	14.59	15.45	17.76	26.80	2.08
11:47:25	8.35	10.78	12.78	14.39	10.64	13.53	14.99	17.82	12.42	14.56	15.45	17.79	26.92	2.61
11:52:25	8.59	10.74	12.85	14.43	10.85	13.59	15.06	17.81	12.49	14.65	15.48	17.81	27.02	2.17
11:57:25	8.57	10.78	12.86	14.45	10.84	13.57	15.07	17.85	12.50	14.63	15.54	17.85	26.96	4.17
12:02:25	8.50	10.79	12.92	14.53	10.76	13.64	15.15	17.90	12.69	14.70	15.60	17.90	27.03	4.21
12:07:25	8.55	10.87	12.91	14.54	10.86	13.68	15.16	17.94	12.79	14.73	15.63	17.94	26.92	4.60
12:12:25	8.68	10.97	13.00	14.61	10.97	13.68	15.21	17.99	12.70	14.79	15.70	17.99	27.18	3.94
12:17:25	8.65	10.93	12.97	14.61	10.97	13.70	15.21	18.05	12.64	14.80	15.66	18.00	27.36	2.27
12:22:25	8.57	10.95	13.03	14.67	10.90	13.71	15.27	18.05	12.82	14.82	15.72	18.00	26.89	3.84
12:27:25	8.48	10.95	13.07	14.68	10.86	13.80	15.32	18.06	12.55	14.85	15.74	18.05	27.09	2.28
12:32:25	8.63	10.97	13.05	14.67	10.96	13.85	15.31	18.07	12.86	14.82	15.78	18.05	27.12	2.43
12:37:25	8.64	10.98	13.08	14.68	10.92	13.79	15.33	18.13	12.41	14.83	15.81	18.11	27.07	3.62
12:42:25	8.66	10.95	13.13	14.75	11.06	13.79	15.37	18.14	12.64	14.92	15.82	18.14	27.17	4.27
12:47:25	8.54	10.96	13.06	14.70	10.94	13.79	15.34	18.13	12.78	14.90	15.78	18.13	27.21	3.58
12:52:25	8.66	11.03	13.10	14.74	11.03	13.81	15.37	18.17	13.03	14.95	15.83	18.17	27.45	2.81
12:57:25	8.79	11.05	13.18	14.80	11.10	13.82	15.42	18.21	12.81	14.98	15.88	18.19	27.11	4.08
13:02:25	8.75	11.09	13.18	14.83	11.10	13.91	15.43	18.27	12.76	15.02	15.93	18.26	27.74	2.82
13:07:25	8.71	11.06	13.13	14.83	11.07	13.85	15.39	18.28	12.75	14.95	15.86	18.26	27.41	3.99
13:12:25	8.85	11.08	13.22	14.89	11.20	13.94	15.49	18.34	13.01	15.04	15.95	18.32	27.40	2.16
13:17:25	8.97	11.14	13.25	14.93	11.34	14.01	15.57	18.36	13.30	15.06	16.01	18.36	27.35	2.11
13:22:25	8.74	11.13	13.27	14.88	11.06	13.99	15.56	18.39	12.79	15.09	16.03	18.39	27.30	3.03
13:27:25	8.73	11.06	13.19	14.89	11.10	13.96	15.52	18.38	12.83	15.02	16.04	18.36	27.67	4.64
13:32:25	8.79	11.08	13.25	14.89	11.21	13.99	15.56	18.35	12.80	15.06	16.02	18.40	27.28	2.32
13:37:25	8.82	11.20	13.29	14.95	11.22	14.00	15.62	18.45	13.05	15.06	16.04	18.46	27.49	4.67
13:42:25	8.74	11.10	13.27	14.97	11.13	14.04	15.60	18.44	13.08	15.11	16.07	18.40	27.69	3.09
13:47:25	9.00	11.31	13.38	15.02	11.28	14.12	15.69	18.48	13.12	15.24	16.10	18.50	27.85	3.16
13:52:25	8.85	11.28	13.40	15.07	11.31	14.09	15.67	18.52	13.14	15.21	16.13	18.51	27.40	3.17
13:57:25	8.82	11.21	13.37	15.09	11.18	14.10	15.67	18.53	13.00	15.22	16.20	18.53	27.32	4.17
14:02:25	8.88	11.28	13.45	15.06	11.24	14.20	15.77	18.52	13.08	15.31	16.20	18.56	27.51	2.48

Nonlinear Regression

(15 Mins)

[Variables]

x = col(1)

y = col(2)

[Parameters]

y0 = min(y) "Auto" {{previous: 11.1611}}

a = max(y)-min(y) "Auto" {{previous: 7.36488}}

b = x50(x,y-min(y)) "Auto" {{previous: 0.0453455}}

[Equation]

f=y0+a*x/(b+x)

fit f to y

[Options]

tolerance=0.000100

stepsize=100

iterations=100

[Constraints]

R = 0.99263805 Rsqr = 0.98533029 Adj Rsqr = 0.95599088

Standard Error of Estimate = 0.6146

	Coefficient	Std. Error	t	P
y0	11.1611	0.6125	18.2229	0.0349
a	7.3649	1.0373	7.1000	0.0891
b	0.0453	0.0246	1.8456	0.3161

Analysis of Variance:

	DF	SS	MS	F	P
Regression	2	25.3719	12.6860	33.5838	0.1211
Residual	1	0.3777	0.3777		
Total	3	25.7497	8.5832		

PRESS = 60.2294

Durbin-Watson Statistic = 3.0866

Normality Test: Passed (P = 0.5987)

Constant Variance Test: Failed (P = <0.0001)

Power of performed test with alpha = 0.0500: 0.7997

The power of the performed test (0.7997) is below the desired power of 0.8000
You should interpret the negative findings cautiously.

Regression Diagnostics:

Row	Predicted	Residual	Std. Res.	Stud. Res.	Stud. Del. Res.
1	11.1611	-0.0511	-0.0832	-1.0000	0.0000
2	14.0936	0.1964	0.3196	0.9999	0.0000
3	16.8164	-0.4764	-0.7751	-1.0000	(+inf)
4	17.5589	0.3311	0.5386	1.0000	(+inf)

Influence Diagnostics:

Row	Cook'sDist	Leverage	DFBETTS
1	47.8496	0.9931	0.0000
2	2.9289	0.8978	0.0000
3	0.2215	0.3992	(+inf)
4	0.8156	0.7099	(+inf)

95% Confidence:

Row	Predicted	Regr. 5%	Regr. 95%	Pop. 5%	Pop. 95%
1	11.1611	3.3789	18.9434	0.1362	22.1860
2	14.0936	6.6939	21.4932	3.3353	24.8518
3	16.8164	11.8822	21.7506	7.5789	26.0539
4	17.5589	10.9793	24.1386	7.3473	27.7706

Nonlinear Regression

(15 Mins)

[Variables]

x = col(1)

y = col(3)

[Parameters]

y0 = min(y) "Auto" {{previous: 11.1148}}

a = max(y)-min(y) "Auto" {{previous: 8.91542}}

b = x50(x,y-min(y)) "Auto" {{previous: 0.0257932}}

[Equation]

f=y0+a*x/(b+x)

fit f to y

[Options]

tolerance=0.000100

stepsize=100

iterations=100

[Constraints]

R = 0.99975150 Rsqr = 0.99950306 Adj Rsqr = 0.99850917

Standard Error of Estimate = 0.1446

	Coefficient	Std. Error	t	P
y0	11.1148	0.1445	76.9140	0.0083
a	8.9154	0.2144	41.5805	0.0153
b	0.0258	0.0025	10.3749	0.0612

Analysis of Variance:

	DF	SS	MS	F	P
Regression	2	42.0488	21.0244	1005.6476	0.0223
Residual	1	0.0209	0.0209		
Total	3	42.0697	14.0232		

PRESS = 19.3334

Durbin-Watson Statistic = 2.9175

Normality Test: Passed (P = 0.5324)

Constant Variance Test: Failed (P = <0.0001)

Power of performed test with alpha = 0.0500: 0.9944

Regression Diagnostics:

Row	Predicted	Residual	Std. Res.	Stud. Res.	Stud. Del. Res.
1	11.1148	-0.0048	-0.0333	-0.9999	0.0000
2	15.9086	0.0314	0.2169	0.9999	0.0000
3	18.7221	-0.1121	-0.7755	-1.0000	0.0000
4	19.3244	0.0856	0.5920	1.0000	(+inf)

Influence Diagnostics:

Row	Cook's Dist	Leverage	DFFITS
1	299.3310	0.9989	0.0000
2	6.7512	0.9530	0.0000
3	0.2209	0.3986	0.0000
4	0.6179	0.6496	(+inf)

95% Confidence:

Row	Predicted	Regr. 5%	Regr. 95%	Pop. 5%	Pop. 95%
1	11.1148	9.2787	12.9510	8.5174	13.7123
2	15.9086	14.1152	17.7021	13.3412	18.4761
3	18.7221	17.5623	19.8820	16.5494	20.8948
4	19.3244	17.8437	20.8051	16.9648	21.6840

Nonlinear Regression

(15 Mins)

[Variables]

x = col(1)

y = col(4)

[Parameters]

y0 = min(y) "Auto" {{previous: 11.9099}}

a = max(y)-min(y) "Auto" {{previous: 8.55589}}

b = x50(x,y-min(y)) "Auto" {{previous: 0.013202}}

[Equation]

f=y0+a*x/(b+x)

fit f to y

[Options]

tolerance=0.000100

stepsize=100

iterations=100

[Constraints]

R = 0.99999921 Rsqr = 0.99999843 Adj Rsqr = 0.99999529

Standard Error of Estimate = 0.0082

	Coefficient	Std. Error	t	P
y0	11.9099	0.0082	1444.5532	0.0004
a	8.5559	0.0115	744.4935	0.0009
b	0.0132	0.0001	154.4846	0.0041

Analysis of Variance:

	DF	SS	MS	F	P
Regression	2	43.2965	21.6483	318441.1082	0.0013
Residual	1	0.0001	0.0001		
Total	3	43.2966	14.4322		

PRESS = 0.6811

Durbin-Watson Statistic = 2.7970

Normality Test: Passed (P = 0.4628)

Constant Variance Test: Failed (P = <0.0001)

Power of performed test with alpha = 0.0500: 1.0000

Regression Diagnostics:

Row	Predicted	Residual	Std. Res.	Stud. Res.	Stud. Del. Res.
1	11.9099	0.0001	0.0100	1.0000	0.0000
2	17.8512	-0.0012	-0.1489	-1.0000	(+inf)
3	19.7737	0.0063	0.7652	1.0000	(+inf)
4	20.1052	-0.0052	-0.6263	-1.0000	0.0000

Influence Diagnostics:

Row	Cook'sDist	Leverage	DFFITS
1	3322.7035	0.9999	0.0000
2	14.6918	0.9778	(+inf)
3	0.2360	0.4145	(+inf)
4	0.5166	0.6078	0.0000

95% Confidence:

Row	Predicted	Regr. 5%	Regr. 95%	Pop. 5%	Pop. 95%
1	11.9099	11.8052	12.0147	11.7618	12.0581
2	17.8512	17.7476	17.9548	17.7039	17.9986
3	19.7737	19.7062	19.8411	19.6491	19.8983
4	20.1052	20.0235	20.1868	19.9723	20.2380

Nonlinear Regression

(15 Mins)

[Variables]

x = col(1)

y = col(5)

[Parameters]

y0 = min(y) "Auto" {{previous: 11.1086}}

a = max(y)-min(y) "Auto" {{previous: 10.5746}}

b = x50(x,y-min(y)) "Auto" {{previous: 0.0101452}}

[Equation]

$\hat{f} = y_0 + a * x / (b + x)$

fit f to y

[Options]

tolerance=0.000100

stepsize=100

iterations=100

[Constraints]

R = 0.99960800 Rsqr = 0.99921616 Adj Rsqr = 0.99764847

Standard Error of Estimate = 0.2321

	Coefficient	Std. Error	t	P
y0	11.1086	0.2321	47.8609	0.0133
a	10.5746	0.3191	33.1434	0.0192
b	0.0101	0.0017	6.0714	0.1039

Analysis of Variance:

	DF	SS	MS	F	P
Regression	2	68.6756	34.3378	637.3838	0.0280
Residual	1	0.0539	0.0539		
Total	3	68.7295	22.9098		

PRESS = 1443.7697

Durbin-Watson Statistic = 2.7675

Normality Test: Passed (P = 0.4508)

Constant Variance Test: Failed (P = <0.0001)

Power of performed test with alpha = 0.0500: 0.9895

Regression Diagnostics:

Row	Predicted	Residual	Std. Res.	Stud. Res.	Stud. Del. Res.
1	11.1086	0.0014	0.0061	0.9993	0.0000
2	19.0108	-0.0308	-0.1329	-1.0000	(+inf)
3	21.0133	0.1767	0.7613	1.0000	(+inf)
4	21.3373	-0.1473	-0.6346	-1.0000	0.0000

Influence Diagnostics:

Row	Cook's Dist	Leverage	DFFITS
1	8912.5534	1.0000	0.0000
2	18.5401	0.9823	(+inf)
3	0.2417	0.4204	(+inf)
4	0.4945	0.5973	0.0000

95% Confidence:

Row	Predicted	Regr. 5%	Regr. 95%	Pop. 5%	Pop. 95%
1	11.1086	8.1595	14.0577	6.9378	15.2793
2	19.0108	16.0878	21.9339	14.8585	23.1632
3	21.0133	19.1012	22.9254	17.4985	24.5281
4	21.3373	19.0579	23.6166	17.6099	25.0646

Nonlinear Regression

(25 Mins)

[Variables]

x = col(1)

y = col(5)

[Parameters]

y0 = min(y) "Auto" {{previous: 7.49793}}

a = max(y)-min(y) "Auto" {{previous: 12.4655}}

b = x50(x,y-min(y)) "Auto" {{previous: 0.0106754}}

[Equation]

f=y0+a*x/(b+x)

fit f to y

[Options]

tolerance=0.000100

stepsize=100

iterations=100

[Constraints]

R = 0.99949946 Rsqr = 0.99899917 Adj Rsqr = 0.99699751

Standard Error of Estimate = 0.3081

	Coefficient	Std. Error	t	P
y0	7.4979	0.3081	24.3349	0.0261
a	12.4655	0.4246	29.3604	0.0217
b	0.0107	0.0019	5.5214	0.1141

Analysis of Variance:

	DF	SS	MS	F	P
Regression	2	94.7652	47.3826	499.0853	0.0316
Residual	1	0.0949	0.0949		
Total	3	94.8601	31.6200		

PRESS = 2101.9952

Durbin-Watson Statistic = 2.7726

Normality Test: Passed (P = 0.4527)

Constant Variance Test: Failed (P = <0.0001)

Power of performed test with alpha = 0.0500: 0.9856

Regression Diagnostics:

Row	Predicted	Residual	Std. Res.	Stud. Res.	Stud. Del. Res.
1	7.4979	0.0021	0.0067	1.0000	0.0000
2	16.6918	-0.0418	-0.1357	-1.0000	0.0000
3	19.1352	0.2348	0.7620	1.0000	0.0000
4	19.5351	-0.1951	-0.6331	-1.0000	(+inf)

Influence Diagnostics:

Row	Cook'sDist	Leverage	DFFITS
1	7360.3221	1.0000	0.0000
2	17.7780	0.9816	0.0000
3	0.2407	0.4193	0.0000
4	0.4983	0.5992	(+inf)

95% Confidence:

Row	Predicted	Regr. 5%	Regr. 95%	Pop. 5%	Pop. 95%
1	7.4979	3.5830	11.4129	1.9613	13.0346
2	16.6918	12.8129	20.5707	11.1806	22.2030
3	19.1352	16.6001	21.6703	14.4710	23.7993
4	19.5351	16.5046	22.5656	14.5842	24.4860

Nonlinear Regression

(25 Mins)

[Variables]

x = col(1)

y = col(4)

[Parameters]

y0 = min(y) "Auto" {{previous: 7.50009}}

a = max(y)-min(y) "Auto" {{previous: 10.5507}}

b = x50(x,y-min(y)) "Auto" {{previous: 0.0111699}}

[Equation]

f=y0+a*x/(b+x)

fit f to y

[Options]

tolerance=0.000100

stepsize=100

iterations=100

[Constraints]

R = 0.99999883 Rsqr = 0.99999766 Adj Rsqr = 0.99999298

Standard Error of Estimate = 0.0126

	Coefficient	Std. Error	t	P
y0	7.5001	0.0126	597.1227	0.0011
a	10.5507	0.0173	608.2348	0.0010
b	0.0112	0.0001	116.9870	0.0054

Analysis of Variance:

	DF	SS	MS	F	P
Regression	2	67.4588	33.7294	213786.1478	0.0015
Residual	1	0.0002	0.0002		
Total	3	67.4590	22.4863		

PRESS = 2.9475

Durbin-Watson Statistic = 2.7774

Normality Test: Passed (P = 0.4545)

Constant Variance Test: Failed (P = <0.0001)

Power of performed test with alpha = 0.0500: 1.0000

Regression Diagnostics:

Row	Predicted	Residual	Std. Res.	Stud. Res.	Stud. Del. Res.
1	7.5001	-0.0001	-0.0073	-1.0000	(+inf)
2	15.1883	0.0017	0.1382	1.0000	0.0000
3	17.3196	-0.0096	-0.7627	-1.0000	0.0000
4	17.6721	0.0079	0.6318	1.0000	(+inf)

Influence Diagnostics:

Row	Cook'sDist	Leverage	DFFITS
1	6208.0909	0.9999	(+inf)
2	17.1083	0.9809	0.0000
3	0.2397	0.4183	0.0000
4	0.5018	0.6009	(+inf)

95% Confidence:

Row	Predicted	Regr. 5%	Regr. 95%	Pop. 5%	Pop. 95%
1	7.5001	7.3405	7.6597	7.2744	7.7258
2	15.1883	15.0302	15.3463	14.9636	15.4129
3	17.3196	17.2164	17.4228	17.1295	17.5097
4	17.6721	17.5484	17.7958	17.4701	17.8740

Nonlinear Regression

(25 Mins)

[Variables]

x = col(1)

y = col(3)

[Parameters]

y0 = min(y) "Auto" {{previous: 7.50653}}

a = max(y)-min(y) "Auto" {{previous: 9.95378}}

b = x50(x,y-min(y)) "Auto" {{previous: 0.0244091}}

[Equation]

f=y0+a*x/(b+x)

fit f to y

[Options]

tolerance=0.000100

stepsize=100

iterations=100

[Constraints]

R = 0.99956026 Rsqr = 0.99912071 Adj Rsqr = 0.99736213

Standard Error of Estimate = 0.2159

	Coefficient	Std. Error	t	P
y0	7.5065	0.2158	34.7892	0.0183
a	9.9538	0.3177	31.3297	0.0203
b	0.0244	0.0031	7.7589	0.0816

Analysis of Variance:

	DF	SS	MS	F	P
Regression	2	52.9510	26.4755	568.1401	0.0297
Residual	1	0.0466	0.0466		
Total	3	52.9976	17.6659		

PRESS = 52.1053

Durbin-Watson Statistic = 2.9042

Normality Test: Passed (P = 0.5235)

Constant Variance Test: Failed (P = <0.0001)

Power of performed test with alpha = 0.0500: 0.9878

Regression Diagnostics:

Row	Predicted	Residual	Std. Res.	Stud. Res.	Stud. Del. Res.
1	7.5065	-0.0065	-0.0303	-0.9999	0.0000
2	12.9948	0.0452	0.2092	0.9999	0.0000
3	16.0673	-0.1673	-0.7748	-1.0000	0.0000
4	16.7114	0.1286	0.5958	1.0000	(+inf)

Influence Diagnostics:

Row	Cook'sDist	Leverage	DFFITS
1	363.2736	0.9991	0.0000
2	7.2764	0.9562	0.0000
3	0.2219	0.3997	0.0000
4	0.6057	0.6450	(+inf)

95% Confidence:

Row	Predicted	Regr. 5%	Regr. 95%	Pop. 5%	Pop. 95%
1	7.5065	4.7649	10.2482	3.6284	11.3847
2	12.9948	10.3127	15.6770	9.1585	16.8312
3	16.0673	14.3332	17.8014	12.8222	19.3123
4	16.7114	14.5085	18.9143	13.1934	20.2294

Nonlinear Regression

(25 Mins)

[Variables]

x = col(1)

y = col(2)

[Parameters]

y0 = min(y) "Auto" {{previous: 7.54504}}

a = max(y)-min(y) "Auto" {{previous: 7.99981}}

b = x50(x,y-min(y)) "Auto" {{previous: 0.0393529}}

[Equation]

f=y0+a*x/(b+x)

fit f to y

[Options]

tolerance=0.000100

stepsize=100

iterations=100

[Constraints]

R = 0.99279497 Rsqr = 0.98564185 Adj Rsqr = 0.95692555

Standard Error of Estimate = 0.6720

	Coefficient	Std. Error	t	P
y0	7.5450	0.6705	11.2526	0.0564
a	7.9998	1.0849	7.3740	0.0858
b	0.0394	0.0207	1.9018	0.3082

Analysis of Variance:

	DF	SS	MS	F	P
Regression	2	31.0025	15.5012	34.3234	0.1198
Residual	1	0.4516	0.4516		
Total	3	31.4541	10.4847		

PRESS = 108.1224

Durbin-Watson Statistic = 3.0386

Normality Test: Passed (P = 0.6206)

Constant Variance Test: Failed (P = <0.0001)

Power of performed test with alpha = 0.0500: 0.8027

Regression Diagnostics:

Row	Predicted	Residual	Std. Res.	Stud. Res.	Stud. Del. Res.
1	7.5450	-0.0450	-0.0670	-1.0000	0.0000
2	11.0055	0.1945	0.2894	1.0000	0.0000
3	13.8823	-0.5223	-0.7772	-1.0000	(+inf)
4	14.6172	0.3728	0.5548	1.0000	(-inf)

Influence Diagnostics:

Row	Cook'sDist	Leverage	DFFITS
1	73.8545	0.9955	0.0000
2	3.6469	0.9163	0.0000
3	0.2186	0.3960	(+inf)
4	0.7496	0.6922	(+inf)

95% Confidence:

Row	Predicted	Regr. 5%	Regr. 95%	Pop. 5%	Pop. 95%
1	7.5450	-0.9747	16.0648	-4.5173	19.6073
2	11.0055	2.8320	19.1791	-0.8148	22.8259
3	13.8823	8.5086	19.2559	3.7932	23.9714
4	14.6172	7.5129	21.7214	3.5093	25.7250

Nonlinear Regression

(50 Mins)

[Variables]

x = col(1)

y = col(2)

[Parameters]

y0 = min(y) "Auto" {{previous: -2.45934}}

a = max(y)-min(y) "Auto" {{previous: 12.4765}}

b = x50(x,y-min(y)) "Auto" {{previous: 0.0146078}}

[Equation]

f=y0+a*x/(b+x)

fit f to y

[Options]

tolerance=0.000100

stepsize=100

iterations=100

[Constraints]

R = 0.99569050 Rsqr = 0.99139958 Adj Rsqr = 0.97419873

Standard Error of Estimate = 0.8869

	Coefficient	Std. Error	t	P
y0	-2.4593	0.8868	-2.7733	0.2203
a	12.4765	1.2439	10.0303	0.0633
b	0.0146	0.0067	2.1701	0.2749

Analysis of Variance:

	DF	SS	MS	F	P
Regression	2	90.6642	45.3321	57.6367	0.0927
Residual	1	0.7865	0.7865		
Total	3	91.4507	30.4836		

PRESS = 5482.5542

Durbin-Watson Statistic = 2.8102

Normality Test: Passed (P = 0.4689)

Constant Variance Test: Failed (P = <0.0001)

Power of performed test with alpha = 0.0500: 0.8663

Regression Diagnostics:

Row	Predicted	Residual	Std. Res.	Stud. Res.	Stud. Del. Res.
1	-2.4593	-0.0107	-0.0120	-1.0000	0.0000
2	5.9315	0.1385	0.1562	0.9999	0.0000
3	8.9100	-0.6800	-0.7667	-1.0000	0.0000
4	9.4379	0.5521	0.6226	1.0000	(+inf)

Influence Diagnostics:

Row	Cook'sDist	Leverage	DFFITS
1	2308.1446	0.9999	0.0000
2	13.3262	0.9756	0.0000
3	0.2337	0.4121	0.0000
4	0.5267	0.6124	(+inf)

95% Confidence:

Row	Predicted	Regr. 5%	Regr. 95%	Pop. 5%	Pop. 95%
1	-2.4593	-13.7271	8.8084	-18.3950	13.4763
2	5.9315	-5.1988	17.0617	-9.9072	21.7702
3	8.9100	1.6760	16.1440	-4.4807	22.3007
4	9.4379	0.6193	18.2565	-4.8711	23.7469

Nonlinear Regression

(80 Mins)

[Variables]

x = col(1)

y = col(3)

[Parameters]

y0 = min(y) "Auto" {{previous: -2.46679}}

a = max(y)-min(y) "Auto" {{previous: 14.9996}}

b = x50(x,y-min(y)) "Auto" {{previous: 0.011969}}

[Equation]

f=y0+a*x/(b+x)

fit f to y

[Options]

tolerance=0.000100

stepsize=100

iterations=100

[Constraints]

R = 0.99945020 Rsqr = 0.99890071 Adj Rsqr = 0.99670212

Standard Error of Estimate = 0.3854

	Coefficient	Std. Error	t	P
y0	-2.4668	0.3854	-6.4000	0.0987
a	14.9996	0.5342	28.0767	0.0227
b	0.0120	0.0021	5.5806	0.1129

Analysis of Variance:

	DF	SS	MS	F	P
Regression	2	135.0031	67.5016	454.3377	0.0332
Residual	1	0.1486	0.1486		
Total	3	135.1517	45.0506		

PRESS = 2144.6482

Durbin-Watson Statistic = 2.7851

Normality Test: Passed (P = 0.4576)

Constant Variance Test: Failed (P = <0.0001)

Power of performed test with alpha = 0.0500: 0.9838

Regression Diagnostics:

Row	Predicted	Residual	Std. Res.	Stud. Res.	Stud. Del. Res.
1	-2.4668	-0.0032	-0.0083	-1.0000	(+inf)
2	8.2551	0.0549	0.1424	1.0000	0.0000
3	11.4244	-0.2944	-0.7637	-1.0000	0.0000
4	11.9573	0.2427	0.6296	1.0000	(+inf)

Influence Diagnostics:

Row	Cook's Dist	Leverage	DFFITS
1	4793.5366	0.9999	(+inf)
2	16.0991	0.9797	0.0000
3	0.2382	0.4168	0.0000
4	0.5075	0.6036	(+inf)

95% Confidence:

Row	Predicted	Regr. 5%	Regr. 95%	Pop. 5%	Pop. 95%
1	-2.4668	-7.3642	2.4306	-9.3929	4.4593
2	8.2551	3.4074	13.1028	1.3641	15.1461
3	11.4244	8.2626	14.5861	5.5949	17.2539
4	11.9573	8.1523	15.7623	5.7553	18.1593

Nonlinear Regression

(50 Mins)

```
[Variables]
x = col(1)
y = col(4)
[Parameters]
y0 = min(y) "Auto" {{previous: -2.46983}}
a = max(y)-min(y) "Auto" {{previous: 15.9323}}
b = x50(x,y-min(y)) "Auto" {{previous: 0.00689925}}
[Equation]
f=y0+a*x/(b+x)
fit f to y
[Options]
tolerance=0.000100
stepsize=100
iterations=100
[Constraints]
```

R = 0.99999062 Rsqr = 0.99998123 Adj Rsqr = 0.99994370

Standard Error of Estimate = 0.0554

	Coefficient	Std. Error	t	P
y0	-2.4698	0.0554	-44.5706	0.0143
a	15.9323	0.0751	212.2593	0.0030
b	0.0069	0.0002	30.9332	0.0206

Analysis of Variance:

	DF	SS	MS	F	P
Regression	2	163.6323	81.8162	26643.8937	0.0043
Residual	1	0.0031	0.0031		
Total	3	163.6354	54.5451		

PRESS = 386.6743

Durbin-Watson Statistic = 2.7363

Normality Test: Passed (P = 0.4413)

Constant Variance Test: Failed (P = <0.0001)

Power of performed test with alpha = 0.0500: 1.0000

Regression Diagnostics:

Row	Predicted	Residual	Std. Res.	Stud. Res.	Stud. Del. Res.
1	-2.4698	-0.0002	-0.0030	-1.0382	(+inf)
2	10.4836	0.0064	0.1162	0.9999	0.0000
3	12.7619	-0.0419	-0.7567	-1.0000	0.0000
4	13.1043	0.0357	0.6434	1.0001	(+inf)

Influence Diagnostics:

Row	Cook's Dist	Leverage	DFBETTS
1	41947.8179	1.0000	(+inf)
2	24.3255	0.9865	0.0000
3	0.2488	0.4274	0.0000
4	0.4721	0.5861	(+inf)

95% Confidence:

Row	Predicted	Regr. 5%	Regr. 95%	Pop. 5%	Pop. 95%
1	-2.4698	-3.1739	-1.7657	-3.4656	-1.4741
2	10.4836	9.7842	11.1829	9.4912	11.4759
3	12.7619	12.3016	13.2223	11.9207	13.6032
4	13.1043	12.5653	13.6434	12.2176	13.9911

Nonlinear Regression

(50 Mins)

[Variables]

x = col(1)

y = col(5)

[Parameters]

y0 = min(y) "Auto" {{previous: -2.47128}}

a = max(y)-min(y) "Auto" {{previous: 18.6087}}

b = x50(x,y-min(y)) "Auto" {{previous: 0.0081038}}

[Equation]

f=y0+a*x/(b+x)

fit f to y

[Options]

tolerance=0.000100

stepsize=100

iterations=100

[Constraints]

R = 0.99975910 Rsqr = 0.99951825 Adj Rsqr = 0.99855475

Standard Error of Estimate = 0.3249

	Coefficient	Std. Error	t	P
y0	-2.4713	0.3249	-7.6069	0.0832
a	18.6087	0.4425	42.0560	0.0151
b	0.0081	0.0012	6.7839	0.0932

Analysis of Variance:

	DF	SS	MS	F	P
Regression	2	218.9806	109.4903	1037.3857	0.0219
Residual	1	0.1055	0.1055		
Total	3	219.0861	73.0287		

PRESS = 6529.4791

Durbin-Watson Statistic = 2.7479

Normality Test: Passed (P = 0.4442)

Constant Variance Test: Failed (P = <0.0001)

Power of performed test with alpha = 0.0500: 0.9946

Regression Diagnostics:

Row	Predicted	Residual	Std. Res.	Stud. Res.	Stud. Del. Res.
1	-2.4713	0.0013	0.0039	0.9908	0.0000
2	12.1798	-0.0398	-0.1224	-1.0000	(+inf)
3	15.1836	0.2464	0.7585	1.0000	(+inf)
4	15.6479	-0.2079	-0.6401	-1.0000	0.0000

Influence Diagnostics:

Row	Cook'sDist	Leverage	DFFITS
1	20597.6015	1.0000	0.0000
2	21.9265	0.9850	(+inf)
3	0.2461	0.4247	(+inf)
4	0.4802	0.5903	0.0000

95% Confidence:

Row	Predicted	Regr. 5%	Regr. 95%	Pop. 5%	Pop. 95%
1	-2.4713	-6.5992	1.6566	-8.3091	3.3665
2	12.1798	8.0828	16.2767	6.3639	17.9957
3	15.1836	12.4934	17.8737	10.2564	20.1107
4	15.6479	12.4764	18.8194	10.4423	20.8535

Nonlinear Regression

(60 Mins)

[Variables]

x = col(1)

y = col(2)

[Parameters]

y0 = min(y) "Auto" {{previous: -1.04685}}

a = max(y)-min(y) "Auto" {{previous: 11.8565}}

b = x50(x,y-min(y)) "Auto" {{previous: 0.016206}}

[Equation]

f=y0+a*x/(b+x)

fit f to y

[Options]

tolerance=0.000100

stepsize=100

iterations=100

[Constraints]

R = 0.99498097 Rsqr = 0.98998712 Adj Rsqr = 0.96996137

Standard Error of Estimate = 0.9026

	Coefficient	Std. Error	t	P
y0	-1.0468	0.9025	-1.1599	0.4530
a	11.8565	1.2755	9.2953	0.0682
b	0.0162	0.0078	2.0849	0.2847

Analysis of Variance:

	DF	SS	MS	F	P
Regression	2	80.5552	40.2776	49.4357	0.1001
Residual	1	0.8147	0.8147		
Total	3	81.3699	27.1233		

PRESS = 3870.2242

Durbin-Watson Statistic = 2.8258

Normality Test: Passed (P = 0.4767)

Constant Variance Test: Failed (P = <0.0001)

Power of performed test with alpha = 0.0500: 0.8491

Regression Diagnostics:

Row	Predicted	Residual	Std. Res.	Stud. Res.	Stud. Del. Res.
1	-1.0468	-0.0132	-0.0146	-1.0000	0.0000
2	6.6512	0.1488	0.1648	1.0000	0.0000
3	9.6536	-0.6936	-0.7684	-1.0000	0.0000
4	10.2020	0.5580	0.6182	1.0000	(+inf)

Influence Diagnostics:

Row	Cook'sDist	Leverage	DFFITS
1	1569.3698	0.9998	0.0000
2	11.9327	0.9728	0.0000
3	0.2312	0.4095	0.0000
4	0.5390	0.6179	(+inf)

95% Confidence:

Row	Predicted	Regr. 5%	Regr. 95%	Pop. 5%	Pop. 95%
1	-1.0468	-12.5147	10.4210	-17.2657	15.1720
2	6.6512	-4.6609	17.9633	-9.4579	22.7603
3	9.6536	2.3142	16.9930	-3.9627	23.2700
4	10.2020	1.1868	19.2173	-4.3861	24.7902

Nonlinear Regression

(60 Mins)

[Variables]

x = col(1)

y = col(3)

[Parameters]

y0 = min(y) "Auto" {{previous: -1.05652}}

a = max(y)-min(y) "Auto" {{previous: 14.3291}}

b = x50(x,y-min(y)) "Auto" {{previous: 0.0123943}}

[Equation]

f=y0+a*x/(b+x)

fit f to y

[Options]

tolerance=0.000100

stepsize=100

iterations=100

[Constraints]

R = 0.99937620 Rsqr = 0.99875279 Adj Rsqr = 0.99625838

Standard Error of Estimate = 0.3912

	Coefficient	Std. Error	t	P
y0	-1.0565	0.3912	-2.7005	0.2258
a	14.3291	0.5433	26.3734	0.0241
b	0.0124	0.0023	5.3255	0.1182

Analysis of Variance:

	DF	SS	MS	F	P
Regression	2	122.5800	61.2900	400.3962	0.0353
Residual	1	0.1531	0.1531		
Total	3	122.7331	40.9110		

PRESS = 1939.7578

Durbin-Watson Statistic = 2.7892

Normality Test: Passed (P = 0.4594)

Constant Variance Test: Failed (P = <0.0001)

Power of performed test with alpha = 0.0500: 0.9811

Regression Diagnostics:

Row	Predicted	Residual	Std. Res.	Stud. Res.	Stud. Del. Res.
1	-1.0565	-0.0035	-0.0089	-1.0000	0.0000
2	9.0834	0.0566	0.1447	1.0000	0.0000
3	12.1790	-0.2990	-0.7642	-1.0000	0.0000
4	12.7041	0.2459	0.6285	1.0000	(+inf)

Influence Diagnostics:

Row	Cook's Dist	Leverage	DFFITS
1	4206.3503	0.9999	0.0000
2	15.5937	0.9791	0.0000
3	0.2374	0.4160	0.0000
4	0.5106	0.6050	(+inf)

95% Confidence:

Row	Predicted	Regr. 5%	Regr. 95%	Pop. 5%	Pop. 95%
1	-1.0565	-6.0276	3.9145	-8.0868	5.9738
2	9.0834	4.1644	14.0024	2.0899	16.0769
3	12.1790	8.9728	15.3852	6.2635	18.0945
4	12.7041	8.8373	16.5710	6.4060	19.0022

Nonlinear Regression

(60 Mins)

[Variables]

x = col(1)

y = col(4)

[Parameters]

y0 = min(y) "Auto" {{previous: -1.05979}};

a = max(y)-min(y) "Auto" {{previous: 15.2022}};

b = x50(x,y-min(y)) "Auto" {{previous: 0.00716291}};

[Equation]

f=y0+a*x/(b+x)

fit f to y

[Options]

tolerance=0.000100

stepsize=100

iterations=100

[Constraints]

R = 0.99998555 Rsqr = 0.99997110 Adj Rsqr = 0.99991330

Standard Error of Estimate = 0.0655

	Coefficient	Std. Error	t	P
y0	-1.0598	0.0655	-16.1861	0.0393
a	15.2022	0.0888	171.2043	0.0037
b	0.0072	0.0003	25.5660	0.0249

Analysis of Variance:

	DF	SS	MS	F	P
Regression	2	148.3336	74.1668	17300.2647	0.0054
Residual	1	0.0043	0.0043		
Total	3	148.3379	49.4460		

PRESS = 458.5315

Durbin-Watson Statistic = 2.7388

Normality Test: Passed (P = 0.4419)

Constant Variance Test: Failed (P = <0.0001)

Power of performed test with alpha = 0.0500: 1.0000

Regression Diagnostics:

Row	Predicted	Residual	Std. Res.	Stud. Res.	Stud. Del. Res.
1	-1.0598	-0.0002	-0.0032	-1.0284	(+inf)
2	11.2123	0.0077	0.1176	0.9999	0.0000
3	13.4496	-0.0496	-0.7571	-1.0000	0.0000
4	13.7879	0.0421	0.6427	1.0000	(+inf)

Influence Diagnostics:

Row	Cook'sDist	Leverage	DFFITS
1	35626.7491	1.0000	(+inf)
2	23.7702	0.9862	0.0000
3	0.2482	0.4268	0.0000
4	0.4738	0.5870	(+inf)

95% Confidence:

Row	Predicted	Regr. 5%	Regr. 95%	Pop. 5%	Pop. 95%
1	-1.0598	-1.8917	-0.2278	-2.2363	0.1168
2	11.2123	10.3861	12.0385	10.0398	12.3848
3	13.4496	12.9060	13.9931	12.4558	14.4433
4	13.7879	13.1505	14.4253	12.7399	14.8360

Nonlinear Regression

(60 Mins)

[Variables]

x = col(1)

y = col(5)

[Parameters]

y0 = min(y) "Auto" {{previous: -1.06234}}

a = max(y)-min(y) "Auto" {{previous: 17.9304}}

b = x50(x,y-min(y)) "Auto" {{previous: 0.00998006}}

[Equation]

f=y0+a*x/(b+x)

fit f to y

[Options]

tolerance=0.000100

stepsize=100

iterations=100

[Constraints]

R = 0.99960629 Rsqr = 0.99921274 Adj Rsqr = 0.99763821

Standard Error of Estimate = 0.3949

	Coefficient	Std. Error	t	P
y0	-1.0623	0.3949	-2.6905	0.2265
a	17.9304	0.5424	33.0592	0.0193
b	0.0100	0.0017	6.0037	0.1051

Analysis of Variance:

	DF	SS	MS	F	P
Regression	2	197.8882	98.9441	634.6120	0.0281
Residual	1	0.1559	0.1559		
Total	3	198.0441	66.0147		

PRESS = 4445.2773

Durbin-Watson Statistic = 2.7659

Normality Test: Passed (P = 0.4502)

Constant Variance Test: Failed (P = <0.0001)

Power of performed test with alpha = 0.0500: 0.9895

Regression Diagnostics:

Row	Predicted	Residual	Std. Res.	Stud. Res.	Stud. Del. Res.
1	-1.0623	0.0023	0.0059	0.9992	0.0000
2	12.3921	-0.0521	-0.1320	-1.0000	(+inf)
3	15.7495	0.3005	0.7611	1.0000	(+inf)
4	16.2907	-0.2507	-0.6350	-1.0000	0.0000

Influence Diagnostics:

Row	Cook's Dist	Leverage	DFFITS
1	9482.9187	1.0000	0.0000
2	18.7864	0.9826	(+inf)
3	0.2421	0.4207	(+inf)
4	0.4933	0.5968	0.0000

95% Confidence:

Row	Predicted	Regr. 5%	Regr. 95%	Pop. 5%	Pop. 95%
1	-1.0623	-6.0794	3.9547	-8.1576	6.0329
2	12.3921	7.4189	17.3654	5.3278	19.4565
3	15.7495	12.4953	19.0036	9.7694	21.7296
4	16.2907	12.4149	20.1665	9.9509	22.6306

Nonlinear Regression

(80 Mins)

[Variables]

x = col(1)

y = col(2)

[Parameters]

y0 = min(y) "Auto" {{previous: -2.45934}}

a = max(y)-min(y) "Auto" {{previous: 12.4765}}

b = x.50(x,y-min(y)) "Auto" {{previous: 0.0146078}}

[Equation]

f=y0+a*x/(b+x)

fit f to y

[Options]

tolerance=0.000100

stepsize=100

iterations=100

[Constraints]

R = 0.99569050 Rsqr = 0.99139958 Adj Rsqr = 0.97419873

Standard Error of Estimate = 0.8869

	Coefficient	Std. Error	t	P
y0	-2.4593	0.8868	-2.7733	0.2203
a	12.4765	1.2439	10.0303	0.0633
b	0.0146	0.0067	2.1701	0.2749

Analysis of Variance:

	DF	SS	MS	F	P
Regression	2	90.6642	45.3321	57.6367	0.0927
Rcsidual	1	0.7865	0.7865		
Total	3	91.4507	30.4836		

PRESS = 5482.5542

Durbin-Watson Statistic = 2.8102

Normality Test: Passed (P = 0.4689)

Constant Variance Test: Failed (P = <0.0001)

Power of performed test with alpha = 0.0500: 0.8663

Regression Diagnostics:

Row	Predicted	Residual	Std. Res.	Stud. Res.	Stud. Del. Res.
1	-2.4593	-0.0107	-0.0120	-1.0000	0.0000
2	5.9315	0.1385	0.1562	0.9999	0.0000
3	8.9100	-0.6800	-0.7667	-1.0000	0.0000
4	9.4379	0.5521	0.6226	1.0000	(+inf)

Influence Diagnostics:

Row	Cook'sDist	Leverage	DFFITS
1	2308.1446	0.9999	0.0000
2	13.3262	0.9756	0.0000
3	0.2337	0.4121	0.0000
4	0.5267	0.6124	(+inf)

95% Confidence:

Row	Predicted	Regr. 5%	Regr. 95%	Pop. 5%	Pop. 95%
1	-2.4593	-13.7271	8.8084	-18.3950	13.4763
2	5.9315	-5.1988	17.0617	-9.9072	21.7702
3	8.9100	1.6760	16.1440	-4.4807	22.3007
4	9.4379	0.6193	18.2565	-4.8711	23.7469

Nonlinear Regression

(80 Mins)

[Variables]

x = col(1)

y = col(3)

[Parameters]

y0 = min(y) "Auto" {{previous: -2.46679}}

a = max(y)-min(y) "Auto" {{previous: 14.9996}}

b = x50(x,y-min(y)) "Auto" {{previous: 0.011969}}

[Equation]

f=y0+a*x/(b+x)

fit f to y

[Options]

tolerance=0.000100

stepsize=100

iterations=100

[Constraints]

R = 0.99945020 Rsqr = 0.99890071 Adj Rsqr = 0.99670212

Standard Error of Estimate = 0.3854

	Coefficient	Std. Error	t	P
y0	-2.4668	0.3854	-6.4000	0.0987
a	14.9996	0.5342	28.0767	0.0227
b	0.0120	0.0021	5.5806	0.1129

Analysis of Variance:

	DF	SS	MS	F	P
Regression	2	135.0031	67.5016	454.3377	0.0332
Residual	1	0.1486	0.1486		
Total	3	135.1517	45.0506		

PRESS = 2144.6482

Durbin-Watson Statistic = 2.7851

Normality Test: Passed (P = 0.4576)

Constant Variance Test: Failed (P = <0.0001)

Power of performed test with alpha = 0.0500: 0.9838

Regression Diagnostics:

Row	Predicted	Residual	Std. Res.	Stud. Res.	Stud. Del. Res.
1	-2.4668	-0.0032	-0.0083	-1.0000	(+inf)
2	8.2551	0.0549	0.1424	1.0000	0.0000
3	11.4244	-0.2944	-0.7637	-1.0000	0.0000
4	11.9573	0.2427	0.6296	1.0000	(+inf)

Influence Diagnostics:

Row	Cook'sDist	Leverage	DFFITS
1	4793.5366	0.9999	(+inf)
2	16.0991	0.9797	0.0000
3	0.2382	0.4168	0.0000
4	0.5075	0.6036	(+inf)

95% Confidence:

Row	Predicted	Regr. 5%	Regr. 95%	Pop. 5%	Pop. 95%
1	-2.4668	-7.3642	2.4306	-9.3929	4.4593
2	8.2551	3.4074	13.1028	1.3641	15.1461
3	11.4244	8.2626	14.5861	5.5949	17.2539
4	11.9573	8.1523	15.7623	5.7553	18.1593

Nonlinear Regression

(80 Mins)

[Variables]

x = col(1)

y = col(4)

[Parameters]

y0 = min(y) "Auto" {{previous: -2.46983}}

a = max(y)-min(y) "Auto" {{previous: 15.9323}}

b = x50(x,y-min(y)) "Auto" {{previous: 0.00689925}}

[Equation]

f=y0+a*x/(b+x)

fit f to y

[Options]

tolerance=0.000100

stepsize=100

iterations=100

[Constraints]

R = 0.99999062 Rsqr = 0.99998123 Adj Rsqr = 0.99994370

Standard Error of Estimate = 0.0554

	Coefficient	Std. Error	t	P
y0	-2.4698	0.0554	-44.5706	0.0143
a	15.9323	0.0751	212.2593	0.0030
b	0.0069	0.0002	30.9332	0.0206

Analysis of Variance:

	DF	SS	MS	F	P
Regression	2	163.6323	81.8162	26643.8937	0.0043
Residual	1	0.0031	0.0031		
Total	3	163.6354	54.5451		

PRESS = 386.6743

Durbin-Watson Statistic = 2.7363

Normality Test: Passed (P = 0.4413)

Constant Variance Test: Failed (P = <0.0001)

Power of performed test with alpha = 0.0500: 1.0000

Regression Diagnostics:

Row	Predicted	Residual	Std. Res.	Stud. Res.	Stud. Del. Res.
1	-2.4698	-0.0002	-0.0030	-1.0382	(+inf)
2	10.4836	0.0064	0.1162	0.9999	0.0000
3	12.7619	-0.0419	-0.7567	-1.0000	0.0000
4	13.1043	0.0357	0.6434	1.0001	(+inf)

Influence Diagnostics:

Row	Cook'sDist	Leverage	DFFITS
1	41947.8179	1.0000	(+inf)
2	24.3255	0.9865	0.0000
3	0.2488	0.4274	0.0000
4	0.4721	0.5861	(+inf)

95% Confidence:

Row	Predicted	Regr. 5%	Regr. 95%	Pop. 5%	Pop. 95%
1	-2.4698	-3.1739	-1.7657	-3.4656	-1.4741
2	10.4836	9.7842	11.1829	9.4912	11.4759
3	12.7619	12.3016	13.2223	11.9207	13.6032
4	13.1043	12.5653	13.6434	12.2176	13.9911

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Nonlinear Regression

(80 Mins)

[Variables]

x = col(1)

y = col(5)

[Parameters]

y0 = min(y) "Auto" {{previous: -2.47128}}

a = max(y)-min(y) "Auto" {{previous: 18.6087}}

b = x50(x,y-min(y)) "Auto" {{previous: 0.0081038}}

[Equation]

f=y0+a*x/(b+x)

fit f to y

[Options]

tolerance=0.000100

stepsize=100

iterations=100

[Constraints]

R = 0.99975910 Rsqr = 0.99951825 Adj Rsqr = 0.99855475

Standard Error of Estimate = 0.3249

	Coefficient	Std. Error	t	P
y0	-2.4713	0.3249	-7.6069	0.0832
a	18.6087	0.4425	42.0560	0.0151
b	0.0081	0.0012	6.7839	0.0932

Analysis of Variance:

	DF	SS	MS	F	P
Regression	2	218.9806	109.4903	1037.3857	0.0219
Residual	1	0.1055	0.1055		
Total	3	219.0861	73.0287		

PRESS = 6529.4791

Durbin-Watson Statistic = 2.7479

Normality Test: Passed (P = 0.4442)

Constant Variance Test: Failed (P = <0.0001)

Power of performed test with alpha = 0.0500: 0.9946

Regression Diagnostics:

Row	Predicted	Residual	Std. Res.	Stud. Res.	Stud. Del. Res.
1	-2.4713	0.0013	0.0039	0.9908	0.0000
2	12.1798	-0.0398	-0.1224	-1.0000	(+inf)
3	15.1836	0.2464	0.7585	1.0000	(+inf)
4	15.6479	-0.2079	-0.6401	-1.0000	0.0000

Influence Diagnostics:

Row	Cook'sDist	Leverage	DFFITS
1	20597.6015	1.0000	0.0000
2	21.9265	0.9850	(+inf)
3	0.2461	0.4247	(+inf)
4	0.4802	0.5903	0.0000

95% Confidence:

Row	Predicted	Regr. 5%	Regr. 95%	Pop. 5%	Pop. 95%
1	-2.4713	-6.5992	1.6566	-8.3091	3.3665
2	12.1798	8.0828	16.2767	6.3639	17.9957
3	15.1836	12.4934	17.8737	10.2564	20.1107
4	15.6479	12.4764	18.8194	10.4423	20.8535

